

**(12) PATENT**  
**(19) AUSTRALIAN PATENT OFFICE**

**(11) Application No. AU 200113566 B2**  
**(10) Patent No. 783245**

**(54) Title**  
**Wellbore casing repair**

**(51)<sup>7</sup> International Patent Classification(s)**  
**E21B 029/10**

**(21) Application No: 200113566**

**(22) Application Date: 2000.10.31**

**(87) WIPO No: WO01/33037**

**(30) Priority Data**

<b>(31) Number</b>	<b>(32) Date</b>	<b>(33) Country</b>
<b>60/162671</b>	<b>1999.11.01</b>	<b>US</b>

**(43) Publication Date : 2001.05.14**

**(43) Publication Journal Date : 2001.07.26**

**(44) Accepted Journal Date : 2005.10.06**

**(71) Applicant(s)**  
**Shell Oil Company**

**(72) Inventor(s)**  
**Robert Lance Cook; David P. Brisco; R. Bruce Stewart; Reece E. Wyant; Lev Ring; James Jang Woo Nahm; Richard Haut; Robert D. Mack; Alan B. Duell; Andrei Gregory Filippov; Kenneth Michael Cowan; William Joseph Dean Dean**

**(74) Agent/Attorney**  
**Davies Collison Cave, Level 15, 1 Nicholson Street, MELBOURNE VIC 3000**

**(56) Related Art**  
**GB 2344606**  
**US 6085838**  
**WO 1999/035368**

(12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

**(19) World Intellectual Property Organization  
International Bureau**



1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100 101 102 103 104 105 106 107 108 109 110 111 112 113 114 115 116 117 118 119 120 121 122 123 124 125 126 127 128 129 130 131 132 133 134 135 136 137 138 139 140 141 142 143 144 145 146 147 148 149 150 151 152 153 154 155 156 157 158 159 160 161 162 163 164 165 166 167 168 169 170 171 172 173 174 175 176 177 178 179 180 181 182 183 184 185 186 187 188 189 190 191 192 193 194 195 196 197 198 199 200 201 202 203 204 205 206 207 208 209 210 211 212 213 214 215 216 217 218 219 220 221 222 223 224 225 226 227 228 229 230 231 232 233 234 235 236 237 238 239 240 241 242 243 244 245 246 247 248 249 250 251 252 253 254 255 256 257 258 259 260 261 262 263 264 265 266 267 268 269 270 271 272 273 274 275 276 277 278 279 280 281 282 283 284 285 286 287 288 289 290 291 292 293 294 295 296 297 298 299 300 301 302 303 304 305 306 307 308 309 310 311 312 313 314 315 316 317 318 319 320 321 322 323 324 325 326 327 328 329 330 331 332 333 334 335 336 337 338 339 340 341 342 343 344 345 346 347 348 349 350 351 352 353 354 355 356 357 358 359 360 361 362 363 364 365 366 367 368 369 370 371 372 373 374 375 376 377 378 379 380 381 382 383 384 385 386 387 388 389 390 391 392 393 394 395 396 397 398 399 400 401 402 403 404 405 406 407 408 409 410 411 412 413 414 415 416 417 418 419 420 421 422 423 424 425 426 427 428 429 430 431 432 433 434 435 436 437 438 439 440 441 442 443 444 445 446 447 448 449 450 451 452 453 454 455 456 457 458 459 460 461 462 463 464 465 466 467 468 469 470 471 472 473 474 475 476 477 478 479 480 481 482 483 484 485 486 487 488 489 490 491 492 493 494 495 496 497 498 499 500 501 502 503 504 505 506 507 508 509 510 511 512 513 514 515 516 517 518 519 520 521 522 523 524 525 526 527 528 529 530 531 532 533 534 535 536 537 538 539 540 541 542 543 544 545 546 547 548 549 550 551 552 553 554 555 556 557 558 559 560 561 562 563 564 565 566 567 568 569 570 571 572 573 574 575 576 577 578 579 580 581 582 583 584 585 586 587 588 589 590 591 592 593 594 595 596 597 598 599 600 601 602 603 604 605 606 607 608 609 610 611 612 613 614 615 616 617 618 619 620 621 622 623 624 625 626 627 628 629 630 631 632 633 634 635 636 637 638 639 640 641 642 643 644 645 646 647 648 649 650 651 652 653 654 655 656 657 658 659 660 661 662 663 664 665 666 667 668 669 670 671 672 673 674 675 676 677 678 679 680 681 682 683 684 685 686 687 688 689 690 691 692 693 694 695 696 697 698 699 700 701 702 703 704 705 706 707 708 709 710 711 712 713 714 715 716 717 718 719 720 721 722 723 724 725 726 727 728 729 730 731 732 733 734 735 736 737 738 739 740 741 742 743 744 745 746 747 748 749 750 751 752 753 754 755 756 757 758 759 760 761 762 763 764 765 766 767 768 769 770 771 772 773 774 775 776 777 778 779 780 781 782 783 784 785 786 787 788 789 790 791 792 793 794 795 796 797 798 799 800 801 802 803 804 805 806 807 808 809 810 811 812 813 814 815 816 817 818 819 820 821 822 823 824 825 826 827 828 829 830 831 832 833 834 835 836 837 838 839 840 841 842 843 844 845 846 847 848 849 850 851 852 853 854 855 856 857 858 859 860 861 862 863 864 865 866 867 868 869 870 871 872 873 874 875 876 877 878 879 880 881 882 883 884 885 886 887 888 889 890 891 892 893 894 895 896 897 898 899 900 901 902 903 904 905 906 907 908 909 910 911 912 913 914 915 916 917 918 919 920 921 922 923 924 925 926 927 928 929 930 931 932 933 934 935 936 937 938 939 940 941 942 943 944 945 946 947 948 949 950 951 952 953 954 955 956 957 958 959 960 961 962 963 964 965 966 967 968 969 970 971 972 973 974 975 976 977 978 979 980 981 982 983 984 985 986 987 988 989 990 991 992 993 994 995 996 997 998 999 1000 1001 1002 1003 1004 1005 1006 1007 1008 1009 1010 1011 1012 1013 1014 1015 1016 1017 1018 1019 1020 1021 1022 1023 1024 1025 1026 1027 1028 1029 1030 1031 1032 1033 1034 1035 1036 1037 1038 1039 1040 1

**(43) International Publication Date**  
**10 May 2001 (10.05.2001)**

**PCT**

**(10) International Publication Number**  
**WO 01/33037 A1**

(51) **International Patent Classification:** E21B 29/10  
(21) **International Application Number:** PCT/US00/30022  
(22) **International Filing Date:** 31 October 2000 (31.10.2000)  
(25) **Filing Language:** English  
(26) **Publication Language:** English  
(30) **Priority Data:**  
60/162,671 1 November 1999 (01.11.1999) US  
(71) **Applicant (for all designated States except US):** SHELL OIL COMPANY [US/US]; 910 Louisiana Street, Houston, TX 77252-2463 (US).

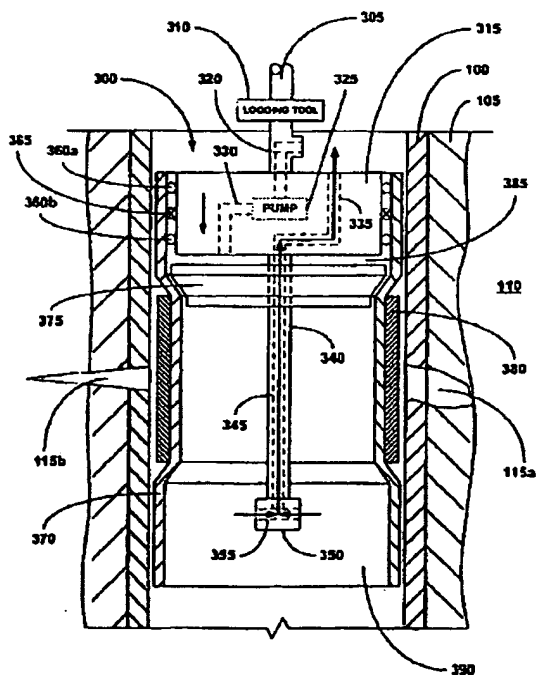
(US). BRISCO, David, P. [US/US]; 405 Westridge Drive, Duncan, OK 73533 (US). STEWART, R., Bruce (GB/NL); Wassenaarweg 208, NL-2596 EC The Hague (NL). WYANT, Reece, E. [US/US]; -. RING, Lev (RU/US); 14126 Heatherhill Place, Houston, TX 77077 (US). NAHM, James, Jang, Woo [-/US]; -. HAUT, Richard [US/US]; 502 Lakebend Drive, Sugar Land, TX 77479-5831 (US). MACK, Robert, D. [US/US]; 22435 Vobe court, Katy, TX 77449 (US). DUELL, Atan, B. [US/US]; 1408 Westeria, Duncan, OK 73533 (US). FILIPPOV, Andrei, Gregory [US/US]; 2606 Hidden Shore Drive, Katy, TX 77450 (US). COWAN, Kenneth, Michael [US/US]; 14211 Lake Trail Drive, Sugar Land, TX 77478 (US). DEAN, William, Joseph, Dean [US/US]; 22602 Crosscreek Cove Court, Katy, TX 77494 (US).

(75) Inventors/Applicants (for US only): COOK, Robert, Lance [US/US]; 934 Caswell Court, Katy, TX 77450

(74) Agents: **MATTINGLY, Tod et al.; Haynes and Boone, LLP**, 901 Main Street, Suite 3100, Dallas, TX 75202-3789 (US).

*(Continued on next page)*

(54) Title: WELLBORE CASING REPAIR



(57) **Abstract:** An apparatus and method for repairing a wellbore casing (100). An opening (115) in a wellbore casing (100) is located using a logging tool (310). An expandable tubular member (370) is then positioned in opposition to the opening (115) in the wellbore casing (100). The expandable tubular member (370) is then radially expanded into intimate contact with the wellbore casing (100).

**WO 01/33037 A1**

- 1 -

## WELLBORE CASING REPAIR

### Background of the Invention

This invention relates generally to repairing and joining tubular members such as wellbore casings, pipelines and tubular structural supports. For convenience only, the invention will be further described with reference to wellbore casings.

Conventionally, when a wellbore is created, a number of casings are installed in the borehole to prevent collapse of the borehole wall and to prevent undesired outflow of drilling fluid into the formation or inflow of fluid from the formation into the borehole. The borehole is drilled in intervals whereby a casing which is to be installed in a lower borehole interval is lowered through a previously installed casing of an upper borehole interval. As a consequence of this procedure the casing of the lower interval is of smaller diameter than the casing of the upper interval. Thus, the casings are in a nested arrangement with casing diameters decreasing in downward direction. Cement annuli are provided between the outer surfaces of the casings and the borehole wall to seal the casings from the borehole wall. As a consequence of this nested arrangement a relatively large borehole diameter is required at the upper part of the wellbore. Such a large borehole diameter involves increased costs due to heavy casing handling equipment, large drill bits and increased volumes of drilling fluid and drill cuttings. Moreover, increased drilling rig time is involved due to required cement pumping, cement hardening, required equipment changes due to large variations in hole diameters drilled in the course of the well, and the large volume of cuttings drilled and removed.

Conventionally, when an opening is formed in the sidewalls of an existing wellbore casing, whether through damage to the casing or because of an intentional perforation of the casing to facilitate production or a fracturing operation, it is often necessary to seal off the opening in the existing wellbore casing. Conventional methods of sealing off such openings are expensive and unreliable.

The present invention is directed to overcoming one or more of the limitations of the existing procedures for repairing and joining tubular members.

### Summary of the Invention

- 2 -

According to a first aspect of the present invention, there is provided a method of repairing an opening in a tubular member, comprising:

positioning an expandable tubular, an expansion cone, and a pump within the tubular member;

- 5 positioning the expandable tubular in opposition to the opening in the tubular member; pressurising an interior portion of the expandable tubular by operating the pump within the tubular member; and
- radially expanding the expandable tubular into intimate contact with the tubular member using the expansion cone.

10 Also according to the first aspect of the invention, there is provided a method of repairing an opening in a tubular member, comprising:

positioning an expandable tubular, an expansion cone, and a pump within the tubular member;

positioning the expandable tubular in opposition to the opening in the tubular member;

- 15 injecting fluidic materials into an interior portion of the expandable tubular using the pump to pressurise the interior portion of the expandable tubular; and
- displacing the expansion cone relative to the expandable tubular to radially expand the expandable tubular into intimate contact with the tubular member..

According to a second aspect of the present invention, there is provided apparatus

20 for repairing a tubular member, comprising:

a support member;

an expandable tubular removably supported by the support member;

an expansion cone movably supported by the support member; and

a pump supported by the support member, the pump being positioned within the

25 expandable tubular and being adapted to pressurise a portion of the interior of the expandable tubular.

Also according to the second aspect of the invention, there is provided apparatus for repairing a tubular member, comprising:

a support member;

30 an expandable tubular removably supported by the support member;

a tubular expansion cone movably supported by the support member; and

- 3 -

a pump positioned within the expandable tubular supported by the support member and adapted to pressurise a portion of the interior of the expandable tubular.

- According to a third aspect of the present invention, there is provided a method of joining a first tubular member to a second tubular member, wherein the outside diameter of the first tubular member is less than the inside diameter of the second tubular member, comprising:
- positioning at least a portion of the first tubular member within the second tubular member;
- positioning a pump within the first tubular member;
- 10 pressurising a portion of the interior of the first tubular member by pumping fluidic materials proximate the first tubular member into the portion of the interior of the first tubular member using the pump; and
- displacing an expansion cone within the interior of the first tubular member to radially expand said at least a portion of the first tubular member into intimate contact with the
- 15 second tubular member.

- Also according to the third aspect of the invention, there is provided a method of joining a first tubular member to a second tubular member, wherein the outside diameter of the first tubular member is less than the inside diameter of the second tubular member, comprising:
- 20 positioning at least a portion of the first tubular member within the second tubular member;
- positioning a pump within the first tubular member;
- pressurising a portion of the interior of the first tubular member by pumping fluidic materials proximate the first tubular member into the portion of the interior of the first
- 25 tubular member by operating the pump; and
- displacing a tubular expansion cone within the interior of the first tubular member to radially expand at least a portion of the first tubular member into intimate contact with the second tubular member.

- Preferred and advantageous features of each aspect of the invention are disclosed in the dependent claims and are described further by way of example only with reference to the accompanying drawings.
- 30

- 3A -

### Brief Description of the Drawings

FIG. 1 is a fragmentary cross-sectional view of a wellbore casing including one or more openings.

FIG. 2 is a flow chart illustration of an embodiment of a method for repairing the  
5 wellbore casing of FIG. 1.

FIG. 3a is a fragmentary cross-sectional view of the placement of an embodiment of a repair apparatus within the wellbore casing of FIG. 1 wherein the expandable tubular member of the apparatus is positioned opposite the openings in the wellbore casing.

FIG. 3b is a fragmentary cross-sectional view of the radial expansion of the  
10 expandable tubular of the apparatus of FIG. 3a.

FIG. 3c is a fragmentary cross-sectional view of the completion of the radial expansion of the expandable tubular of the apparatus of FIG. 3b.

FIG. 3d is a fragmentary cross-sectional view of the removal of the repair apparatus from the repaired wellbore casing of FIG. 3c.

FIG. 3e is a fragmentary cross-sectional view of the repaired wellbore casing of  
15 FIG. 3d.

FIG. 4 is a cross-sectional illustration of an embodiment of the expandable tubular of the apparatus of FIG. 3a.

FIG. 5 is a flow chart illustration of an embodiment of a method for fabricating the  
20 expandable tubular of the apparatus of FIG. 3a.

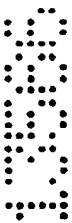


FIG. 6 is a fragmentary cross-sectional illustration of a preferred embodiment of the expandable tubular of FIG. 4.

FIG. 7 is a fragmentary cross-sectional illustration of an expansion cone expanding a tubular member.

5        FIG. 8 is a graphical illustration of the relationship between propagation pressure and the angle of attack of the expansion cone.

FIG. 9 is an illustration of an embodiment of an expansion cone optimally adapted to radially expand the expandable tubular member of FIG. 4.

10       FIG. 10 is an illustration of another embodiment of an expansion cone optimally adapted to radially expand the expandable tubular member of FIG. 4.

FIG. 11 is a fragmentary cross-sectional illustration of the lubrication of the interface between an expansion cone and a tubular member during the radial expansion process.

15       FIG. 12 is an illustration of an embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 13 is an illustration of another embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

20       FIG. 14 is an illustration of another embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 15 is an illustration of another embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a 25 tubular member during the radial expansion of the tubular member.

FIG. 16 is an illustration of another embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

30       FIG. 17 is an illustration of another embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 18 is an illustration of another embodiment of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 19 is an illustration of a preferred embodiment of an expansion cone 5 including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 20 is a cross-sectional illustration of the first axial groove of the expansion cone of FIG. 19.

FIG. 21 is a cross-sectional illustration of the circumferential groove of the 10 expansion cone of FIG. 19.

FIG. 22 is a cross-sectional illustration of one of the second axial grooves of the expansion cone of FIG. 19.

FIG. 23 is a cross sectional illustration of an embodiment of an expansion cone including internal flow passages having inserts for adjusting the flow of 15 lubricant fluids.

FIG. 24 is a cross sectional illustration of the expansion cone of FIG. 23 further including an insert having a filter for filtering out foreign materials from the lubricant fluids.

FIG. 25 is a fragmentary cross sectional illustration of an embodiment of the 20 expansion cone of the repair apparatus of FIG. 3a.

FIG. 26a is a fragmentary cross-sectional view of the placement of another embodiment of a repair apparatus within the wellbore casing of FIG. 1 wherein the expandable tubular member of the apparatus is positioned opposite the openings in the wellbore casing.

FIG. 26b is a fragmentary cross-sectional view of the radial expansion of the 25 expandable tubular of the apparatus of FIG. 26a.

FIG. 26c is a fragmentary cross-sectional view of the completion of the radial expansion of the expandable tubular of the apparatus of FIG. 26b.

FIG. 26d is a fragmentary cross-sectional view of the removal of the repair 30 apparatus from the repaired wellbore casing of FIG. 26c.

FIG. 26e is a fragmentary cross-sectional view of the repaired wellbore casing of FIG. 26d.



### Detailed Description of the Illustrative Embodiments

An apparatus and method for repairing a wellbore casing within a subterranean formation is provided. The apparatus and method permits a wellbore casing to be repaired in a subterranean formation by placing a tubular member, an expansion cone, and a pump in an existing section of a wellbore, and then extruding the tubular member off of the expansion cone by pressurizing an interior portion of the tubular member using the pump. The apparatus and method further permits adjacent tubular members in the wellbore to be joined using an overlapping joint that prevents fluid and or gas passage. The apparatus and method further permits a new tubular member to be supported by an existing tubular member by expanding the new tubular member into engagement with the existing tubular member. The apparatus and method further minimizes the reduction in the hole size of the wellbore casing necessitated by the addition of new sections of wellbore casing. The apparatus and method provide an efficient and reliable method for forming and repairing wellbore casings, pipelines, and structural supports.

The apparatus and method preferably further includes a lubrication and self-cleaning system for the expansion cone. In a preferred implementation, the expansion cone includes one or more circumferential grooves and one or more axial grooves for providing a supply of lubricating fluid to the trailing edge portion of the interface between the expansion cone and a tubular member during the radial expansion process. In this manner, the frictional forces created during the radial expansion process are reduced which results in a reduction in the required operating pressures for radially expanding the tubular member. Furthermore, the supply of lubricating fluid preferably removes loose material from tapered end of the expansion cone that is formed during the radial expansion process.

The apparatus and method preferably further includes an expandable tubular member that includes pre-expanded ends. In this manner, the subsequent radial expansion of the expandable tubular member is optimized.

The apparatus and method preferably further includes an expansion cone for expanding the tubular member includes a first outer surface having a first angle of attack and a second outer surface having a second angle of attack less than the first

angle of attack. In this manner, the expansion of tubular members is optimally provided.

In several alternative embodiments, the apparatus and methods are used to form and/or repair wellbore casings, pipelines, and/or structural supports.

5 Referring initially to FIG. 1, a wellbore casing 100 having an outer annular layer 105 of a sealing material is positioned within a subterranean formation 110. The wellbore casing 100 may be positioned in any orientation from vertical to horizontal. The wellbore casing 100 further includes one or more openings 115a and 115b. The openings 115 may, for example, be the result of: defects in the  
10 wellbore casing 100, intentional perforations of the casing to facilitate production, thin walled sections of casing caused by drilling and/or wireline wear, or fracturing operations. As will be recognized by persons having ordinary skill in the art, such openings 115 in a wellbore 100 can seriously adversely impact the subsequent production of oil and gas from the subterranean formation 110 unless they are  
15 sealed off. More generally, the wellbore casing 115 may include thin walled sections that need cladding in order to prevent a catastrophic failure.

Referring to FIG. 2, a preferred embodiment of a method 200 for repairing a defect in a wellbore casing using a repair apparatus having a logging tool, a pump, an expansion cone, and an expandable tubular member includes the steps of: (1)  
20 positioning the repair apparatus within the wellbore casing in step 205; (2) locating the defect in the wellbore casing using the logging tool of the repair apparatus in step 210; (3) positioning the expandable tubular member in opposition to the defect in the wellbore casing in step 215; and (4) radially expanding the expandable tubular member into intimate contact with the wellbore casing by pressurizing a  
25 portion of the expandable tubular member using the pump and extruding the expandable tubular member off of the expansion cone in step 220. In this manner, defects in a wellbore casing are repaired by a compact and self-contained repair apparatus that is positioned downhole. More generally, the repair apparatus is used to repair defects in wellbore casings, pipelines, and structural supports.

30 As illustrated in FIG. 3a, in a preferred embodiment, in step 205, a repair apparatus 300 is positioned within the wellbore casing 100.

In a preferred embodiment, the repair apparatus 300 includes a first support member 305, a logging tool 310, a housing 315, a first fluid conduit 320, a pump 325, a second fluid conduit 330, a third fluid conduit 335, a second support member 340, a fourth fluid conduit 345, a third support member 350, a fifth fluid conduit 355, 5 sealing members 360, a locking member 365, an expandable tubular 370, an expansion cone 375, and a sealing member 380.

The first support member 305 is preferably coupled to the logging tool 310 and the housing 315. The first support member 305 is preferably adapted to be coupled to and supported by a conventional support member such as, for example, 10 a wireline, coiled tubing, or a drill string. The first support member 305 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials from the repair apparatus 300. The first support member 305 is further preferably adapted to convey electrical power and communication signals to the logging tool 310, the pump 325, and the locking 15 member 365.

The logging tool 310 is preferably coupled to the first support member 305. The logging tool 310 is preferably adapted to detect defects in the wellbore casing 100. The logging tool 310 may be any number of conventional commercially available logging tools suitable for detecting defects in wellbore casings, pipelines, 20 or structural supports. In a preferred embodiment, the logging tool 310 is a CAST logging tool, available from Halliburton Energy Services in order to optimally provide detection of defects in the wellbore casing 100. In a preferred embodiment, the logging tool 310 is contained within the housing 315 in order to provide an repair apparatus 300 that is rugged and compact.

25 The housing 315 is preferably coupled to the first support member 305, the second support member 340, the sealing members 360, and the locking member 365. The housing 315 is preferably releasably coupled to the tubular member 370. The housing 315 is further preferably adapted to contain and/or support the logging tool 310 and the pump 325.

30 The first fluid conduit 320 is preferably fluidically coupled to the inlet of the pump 325 and the exterior region above the housing 315. The first fluid conduit 320 may be contained within the first support member 305 and the housing 315. The

first fluid conduit 320 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

5       The pump 325 is fluidically coupled to the first fluid conduit 320 and the second fluid conduit 330. The pump 325 is further preferably contained within and supported by the housing 315. Alternatively, the pump 325 may be positioned above the housing 315. The pump 325 is preferably adapted to convey fluidic materials from the first fluid conduit 320 to the second fluid conduit 330 at operating  
10 pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally provide the operating pressure for propagating the expansion cone 375. The pump 325 may be any number of conventional commercially available pumps. In a preferred embodiment, the pump 325 is a flow control pump out section for dirty fluids, available from Halliburton Energy  
15 Services in order to optimally provide the operating pressures and flow rates for propagating the expansion cone 375. The pump 325 is preferably adapted to pressurize an interior portion 385 of the expandable tubular member 370 to operating pressures ranging from about 0 to 12,000 psi.

      The second fluid conduit 330 is fluidically coupled to the outlet of the pump 325  
20 and the interior portion 385 of the expandable tubular member 370. The second fluid conduit 330 is further preferably contained within the housing 315. The second fluid conduit 330 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to  
25 optimally propagate the expansion cone 375.

      The third fluid conduit 335 is fluidically coupled to the exterior region above the housing 315 and the interior portion 385 of the expandable tubular member 370. The third fluid conduit 335 is further preferably contained within the housing 315. The third fluid conduit 330 is preferably adapted to convey fluidic materials such as,  
30 for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The second support member 340 is coupled to the housing 315 and the third support member 350. The second support member 340 is further preferably movably and sealingly coupled to the expansion cone 375. The second support member 340 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials. In a preferred embodiment, the second support member 340 is centrally positioned within the expandable tubular member 370.

The fourth fluid conduit 345 is fluidically coupled to the third fluid conduit 335 and the fifth fluid conduit 355. The fourth fluid conduit 345 is further preferably contained within the second support member 340. The fourth fluid conduit 345 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The third support member 350 is coupled to the second support member 340. The third support member 350 is further preferably adapted to support the expansion cone 375. The third support member 350 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials.

The fifth fluid conduit 355 is fluidically coupled to the fourth fluid conduit 345 and a portion 390 of the expandable tubular member 375 below the expansion cone 375. The fifth fluid conduit 355 is further preferably contained within the third support member 350. The fifth fluid conduit 355 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The sealing members 360 are preferably coupled to the housing 315. The sealing members 360 are preferably adapted to seal the interface between the exterior surface of the housing 315 and the interior surface of the expandable tubular member 370. In this manner, the interior portion 385 of the expandable tubular member 375 is fluidically isolated from the exterior region above the housing 315. The sealing members 360 may be any number of conventional commercially

available sealing members. In a preferred embodiment, the sealing members 360 are conventional O-ring sealing members available from various commercial suppliers in order to optimally provide a high pressure seal.

The locking member 365 is preferably coupled to the housing 315. The locking member 365 is further preferably releasably coupled to the expandable tubular member 370. In this manner, the housing 365 is controllably coupled to the expandable tubular member 370. In this manner, the housing 365 is preferably released from the expandable tubular member 370 upon the completion of the radial expansion of the expandable tubular member 370. The locking member 365 may be any number of conventional commercially available releasable locking members. In a preferred embodiment, the locking member 365 is an electrically releasable locking member in order to optimally provide an easily retrievable running expansion system.

In an alternative embodiment, the locking member 365 is replaced by or supplemented by one or more conventional shear pins in order to provide an alternative means of controllably releasing the housing 315 from the expandable tubular member 370.

The expandable tubular member 370 is releasably coupled to the locking member 365. The expandable tubular member 370 is preferably adapted to be radially expanded by the axial displacement of the expansion cone 375.

In a preferred embodiment, as illustrated in FIG. 4, the expandable tubular member 370 includes a tubular body 405 having an interior region 410, an exterior surface 415, a first end 420, an intermediate portion 425, and a second end 430. The tubular member 370 further preferably includes the sealing member 380.

The tubular body 405 of the tubular member 370 preferably has a substantially annular cross section. The tubular body 405 may be fabricated from any number of conventional commercially available materials such as, for example, Oilfield Country Tubular Goods (OCTG), 13 chromium steel, 4140 steel, or automotive grade steel tubing/casing, or L83, J55, or P110 API casing. In a preferred embodiment, the tubular body 405 of the tubular member 370 is further provided substantially as disclosed in one or more of the following co-pending U.S. patent applications:

	Provisional Patent Application Number	Attorney Docket No.	Filing Date
	60/108,558	25791.9	11-16-1998
	60/111,293	25791.3	12-7-1998
5	60/119,611	25791.8	2-11-1999
	60/121,702	25791.7	2-25-1999
	60/121,841	25791.12	2-26-1999
	60/121,907	25791.16	2-26-1999
	60/124,042	25791.11	3-11-1999
10	60/131,106	25791.23	4-26-1999
	60/137,998	25791.17	6-7-1999
	60/143,039	25791.26	7-9-1999
	60/146,203	25791.25	7-29-1999
	60/154,047	25791.29	9-16-1999
15	60/159,082	25791.34	10-12-1999
	60/159,039	25791.36	10-12-1999
	60/159,033	25791.37	10-12-1999

Applicants incorporate by reference the disclosures of these applications.

The interior region 410 of the tubular body 405 preferably has a substantially  
 20 circular cross section. The interior region 410 of the tubular body 405 preferably  
 includes a first inside diameter  $D_1$ , an intermediate inside diameter  $D_{INT}$ , and a  
 second inside diameter  $D_2$ . In a preferred embodiment, the first and second inside  
 diameters,  $D_1$  and  $D_2$ , are substantially equal. In a preferred embodiment, the first  
 and second inside diameters,  $D_1$  and  $D_2$ , are greater than the intermediate inside  
 25 diameter  $D_{INT}$ .

The first end 420 of the tubular body 405 is coupled to the intermediate  
 portion 425 of the tubular body 405. The exterior surface of the first end 420 of the  
 tubular body 405 preferably further includes a protective coating fabricated from  
 tungsten carbide, or other similar wear resistant materials in order to protect the

first end 420 of the tubular body 405 during placement of the repair apparatus 300 within the wellbore casing 100. In a preferred embodiment, the outside diameter of the first end 420 of the tubular body 405 is greater than the outside diameter of the intermediate portion 425 of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. In a preferred embodiment, the outside diameter of the first end 420 of the tubular body 405 is substantially equal to the outside diameter of the second end 430 of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. In a preferred embodiment, the outside diameter of the first end 420 of the tubular member 370 is adapted to permit insertion of the tubular member 370 into the typical range of wellbore casings. The first end 420 of the tubular member 370 includes a wall thickness  $t_1$ .

The intermediate portion 425 of the tubular body 405 is coupled to the first end 420 of the tubular body 405 and the second end 430 of the tubular body 405. The intermediate portion 425 of the tubular body 405 preferably includes the sealing member 380. In a preferred embodiment, the outside diameter of the intermediate portion 425 of the tubular body 405 is less than the outside diameter of the first and second ends, 420 and 430, of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. In a preferred embodiment, the outside diameter of the intermediate portion 425 of the tubular body 405 ranges from about 75% to 98% of the outside diameters of the first and second ends, 420 and 430, in order to optimally protect the sealing member 380 during placement of the tubular member 370 within the wellbore casing 100. The intermediate portion 425 of the tubular body 405 includes a wall thickness  $t_{INT}$ .

The second end 430 of the tubular body 405 is coupled to the intermediate portion 425 of the tubular body 405. The exterior surface of the second end 430 of the tubular body 405 preferably further includes a protective coating fabricated from a wear resistant material such as, for example, tungsten carbide in order to protect the second end 430 of the tubular body 405 during placement of the repair apparatus 300 within the wellbore casing 100. In a preferred embodiment, the



outside diameter of the second end 430 of the tubular body 405 is greater than the outside diameter of the intermediate portion 425 of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within a wellbore casing 100. In a preferred embodiment, the outside diameter of the second end 430 of the tubular body 405 is substantially equal to the outside diameter of the first end 420 of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. In a preferred embodiment, the outside diameter of the second end 430 of the tubular member 370 is adapted to permit insertion of the tubular member 370 into the typical range of wellbore casings. The second end 430 of the tubular member 370 includes a wall thickness  $t_2$ .

In a preferred embodiment, the wall thicknesses  $t_1$  and  $t_2$  are substantially equal in order to provide substantially equal burst strength for the first and second ends, 420 and 430, of the tubular member 370. In a preferred embodiment, the wall thicknesses  $t_1$  and  $t_2$  are both greater than the wall thickness  $t_{INT}$  in order to optimally match the burst strength of the first and second ends, 420 and 430, of the tubular member 370 with the intermediate portion 425 of the tubular member 370.

The sealing member 380 is preferably coupled to the outer surface of the intermediate portion 425 of the tubular body 405. The sealing member 380 preferably seals the interface between the intermediate portion 425 of the tubular body 405 and interior surface of the wellbore casing 100 after radial expansion of the intermediate portion 425 of the tubular body 405. The sealing member 380 preferably has a substantially annular cross section. The outside diameter of the sealing member 380 is preferably selected to be less than the outside diameters of the first and second ends, 420 and 430, of the tubular body 405 in order to optimally protect the sealing member 380 during placement of the tubular member 370 within the typical range of wellbore casings 100. The sealing member 380 may be fabricated from any number of conventional commercially available materials such as, for example, thermoset or thermoplastic polymers. In a preferred embodiment, the sealing member 380 is fabricated from thermoset polymers in order to optimally

seal the interface between the radially expanded intermediate portion 425 of the tubular body 405 and the wellbore casing 100.

During placement of the tubular member 370 within the wellbore casing 100, the protective coatings provided on the exterior surfaces of the first and second ends, 420 and 430, of the tubular body 405 prevent abrasion with the interior surface of the wellbore casing 100. In a preferred embodiment, after radial expansion of the tubular body 405, the sealing member 380 seals the interface between the outside surface of the intermediate portions 425 of the tubular body 405 of the tubular member 370 and the inside surface of the wellbore casing 100. During placement of the tubular member 370 within the wellbore casing 100, the sealing member 380 is preferably protected from contact with the interior walls of the wellbore casing 100 by the recessed outer surface profile of the tubular member 370.

In a preferred embodiment, the tubular body 405 of the tubular member 370 further includes first and second transition portions, 435 and 440, coupled between the first and second ends, 420 and 430, and the intermediate portion 425 of the tubular body 405. In a preferred embodiment, the first and second transition portions, 435 and 440, are inclined at an angle,  $\alpha$ , relative to the longitudinal direction ranging from about 0 to 30 degrees in order to optimally facilitate the radial expansion of the tubular member 370. In a preferred embodiment, the first and second transition portions, 435 and 440, provide a smooth transition between the first and second ends, 420 and 440, and the intermediate portion 425, of the tubular body 405 of the tubular member 370 in order to minimize stress concentrations.

Referring to FIG. 5, in a preferred embodiment, the tubular member 370 is formed by a process 500 that includes the steps of: (1) expanding both ends of the tubular body 405 in step 505; (2) stress relieving both radially expanded ends of the tubular body 405 in step 510; and (3) putting a sealing material on the outside diameter of the non-expanded intermediate portion 425 of the tubular body 405 in step 515. In an alternative embodiment, the process 500 further includes the step of putting layers of protective coatings onto the exterior surfaces of the radially expanded ends, 420 and 430, of the tubular body 405.

In a preferred embodiment, in steps 505 and 510, both ends, 420 and 430, of the tubular body 405 are radially expanded using conventional radial expansion methods, and then both ends, 420 and 430, of the tubular body 405 are stress relieved. The radially expanded ends, 420 and 430, of the tubular body 405 include 5 interior diameters  $D_1$  and  $D_2$ . In a preferred embodiment, the interior diameters  $D_1$  and  $D_2$  are substantially equal in order to provide a burst strength that is substantially equal. In a preferred embodiment, the ratio of the interior diameters  $D_1$  and  $D_2$  to the interior diameter  $D_{INT}$  of the tubular body 405 ranges from about 100% to 120% in order to optimally provide a tubular member for subsequent radial 10 expansion.

In a preferred embodiment, the relationship between the wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  of the tubular body 405; the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$  of the tubular body 405; the inside diameter  $D_{wellbore}$  of the wellbore casing 100 that the tubular body 405 will be inserted into; and the outside diameter  $D_{cone}$  of the 15 expansion cone 375 that will be used to radially expand the tubular body 405 within the wellbore casing 100 is given by the following expression:

$$D_{wellbore} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} \left[ (t_1 - t_{INT}) * D_{cone} + t_{INT} * D_{INT} \right] \quad (1)$$

where  $t_1 = t_2$ ; and

$$D_1 = D_2$$

20 By satisfying the relationship given in equation (1), the expansion forces placed upon the tubular body 405 during the subsequent radial expansion process are substantially equalized. More generally, the relationship given in equation (1) may be used to calculate the optimal geometry for the tubular body 405 for subsequent radial expansion of the tubular body 405 for fabricating and/or repairing a wellbore 25 casing, a pipeline, or a structural support.

In a preferred embodiment, in step 515, the sealing member 380 is then applied onto the outside diameter of the non-expanded intermediate portion 425 of the tubular body 405. The sealing member 380 may be applied to the outside diameter of the non-expanded intermediate portion 425 of the tubular body 405 30 using any number of conventional commercially available methods. In a preferred

embodiment, the sealing member 380 is applied to the outside diameter of the intermediate portion 425 of the tubular body 405 using commercially available chemical and temperature resistant adhesive bonding.

In a preferred embodiment, as illustrated in FIG. 6, the interior surface of the tubular body 405 of the tubular member 370 further includes a coating 605 of a lubricant. The coating 605 of lubricant may be applied using any number of conventional methods such as, for example, dipping, spraying, sputter coating or electrostatic deposition. In a preferred embodiment, the coating 605 of lubricant is chemically, mechanically, and/or adhesively bonded to the interior surface of the tubular body 405 of the tubular member 370 in order to optimally provide a durable and consistent lubricating effect. In a preferred embodiment, the force that bonds the lubricant to the interior surface of the tubular body 405 of the tubular member 370 is greater than the shear force applied during the radial expansion process.

In a preferred embodiment, the coating 605 of lubricant is applied to the interior surface of the tubular body 405 of the tubular member 370 by first applying a phenolic primer to the interior surface of the tubular body 405 of the tubular member 370, and then bonding the coating 605 of lubricant to the phenolic primer using an antifriction paste including the coating 605 of lubricant carried within an epoxy resin. In a preferred embodiment, the antifriction paste includes, by weight, 40-80% epoxy resin, 15-30% molybdenum disulfide, 10-15% graphite, 5-10% aluminum, 5-10% copper, 8-15% aluminosilicate, and 5-10% polyethylenepolyamine. In a preferred embodiment, the antifriction paste is provided substantially as disclosed in U.S. Patent No. 4,329,238, the disclosure of which is incorporated herein by reference.

The coating 605 of lubricant may be any number of conventional commercially available lubricants such as, for example, metallic soaps or zinc phosphates. In a preferred embodiment, the coating 605 of lubricant includes C-Lube-10, C-Phos-52, C-Phos-58-M, and/or C-Phos-58-R in order to optimally provide a coating of lubricant. In a preferred embodiment, the coating 605 of lubricant provides a sliding coefficient of friction less than about 0.20 in order to optimally reduce the force required to radially expand the tubular member 370 using the expansion cone 375.

In an alternative embodiment, the coating 605 includes a first part of a lubricant. In a preferred embodiment, the first part of the lubricant forms a first part of a metallic soap. In an preferred embodiment, the first part of the lubricant coating includes zinc phosphate. In a preferred embodiment, the second part of the  
5 lubricant is circulated within a fluidic carrier that is circulated into contact with the coating 605 of the first part of the lubricant during the radial expansion of the tubular member 370. In a preferred embodiment, the first and second parts of the lubricant react to form a lubricating layer between the interior surface of the tubular body 405 of the tubular member 370 and the exterior surface of the  
10 expansion cone 375 during the radial expansion process. In this manner, a lubricating layer is optimally provided in the exact concentration, exactly when and where it is needed. Furthermore, because the second part of the lubricant is circulated in a carrier fluid, the dynamic interface between the interior surface of the tubular body 405 of the tubular members 370 and the exterior surface of the  
15 expansion cone 375 is also preferably provided with hydrodynamic lubrication. In a preferred embodiment, the first and second parts of the lubricant react to form a metallic soap. In a preferred embodiment, the second part of the lubricant is sodium stearate.

The expansion cone 375 is movably coupled to the second support member  
20 340. The expansion cone 375 is preferably adapted to be axially displaced upon the pressurization of the interior region 385 of the expandable tubular member 370. The expansion cone 375 is further preferably adapted to radially expand the expandable tubular member 370.

In a preferred embodiment, as illustrated in FIG. 7, the expansion cone 375  
25 includes a conical outer surface 705 for radially expanding the tubular member 370 having an angle of attack  $\alpha$ . In a preferred embodiment, as illustrated in FIG. 8, the angle of attack  $\alpha$  ranges from about 10 to 40 degrees in order to minimize the required operating pressure of the interior portion 385 during the radial expansion process.

30 Referring to FIG. 9, an alternative preferred embodiment of an expansion cone 900 for use in the repair apparatus 300 includes a front end 905, a rear end 910, and a radial expansion section 915. In a preferred embodiment, when the

expansion cone 900 is displaced in the longitudinal direction relative to the tubular member 370, the interaction of the exterior surface of the radial expansion section 915 with the interior surface of the tubular member 370 causes the tubular member 370 to expand in the radial direction.

5       The radial expansion section 915 preferably includes a leading radial expansion section 920 and a trailing radial expansion section 925. In a preferred embodiment, the leading and trailing radial expansion sections, 920 and 925, have substantially conical outer surfaces. In a preferred embodiment, the leading and trailing radial expansion sections, 920 and 925, have corresponding angles of attack,  
10  $\alpha_1$  and  $\alpha_2$ . In a preferred embodiment, the angle of attack  $\alpha_1$  of the leading radial expansion section 920 is greater than the angle of attack  $\alpha_2$  of the trailing radial expansion section 925 in order to optimize the radial expansion of the tubular member 370. More generally, the radial expansion section 915 may include one or more intermediate radial expansion sections positioned between the leading and  
15 trailing radial expansion sections, 920 and 925, wherein the corresponding angles of attack  $\alpha$  increase in stepwise fashion from the leading radial expansion section 920 to the trailing radial expansion section 925.

Referring to FIG. 10, another alternative preferred embodiment of an expansion cone 1000 for use in the repair apparatus 300 includes a front end 1005,  
20 a rear end 1010, and a radial expansion section 1015. In a preferred embodiment, when the expansion cone 1000 is displaced in the longitudinal direction relative to the tubular member 370, the interaction of the exterior surface of the radial expansion section 1015 with the interior surface of the tubular member 370 causes the tubular member 370 to expand in the radial direction.

25       The radial expansion section 1015 preferably includes an outer surface 1020 having a substantially parabolic outer profile. In this manner, the outer surface 1020 provides an angle of attack that constantly decreases from a maximum at the front end 1005 of the expansion cone 1000 to a minimum at the rear end 1010 of the expansion cone 1000. The parabolic outer profile of the outer surface 1020 may be  
30 formed using a plurality of adjacent discrete conical sections and/or using a continuous curved surface. In this manner, the area of the outer surface 1020 adjacent to the front end 1005 of the expansion cone 1000 optimally radially

overexpands the intermediate portion 425 of the tubular body 405 of the tubular member 370, while the area of the outer surface 1020 adjacent to the rear end 1010 of the expansion cone 1000 optimally radially overexpands the pre-expanded first and second ends, 420 and 430, of the tubular body 405 of the tubular member 370.

5 In a preferred embodiment, the parabolic profile of the outer surface 1020 is selected to provide an angle of attack that ranges from about 8 to 20 degrees in the vicinity of the front end 1005 of the expansion cone 1000 and an angle of attack in the vicinity of the rear end 1010 of the expansion cone 1000 from about 4 to 15 degrees.

Referring to FIG. 11, the lubrication of the interface between the expansion  
10 cone 370 and the tubular member 375 during the radial expansion process will now be described. As illustrated in FIG. 31, during the radial expansion process, an expansion cone 370 radially expands the tubular member 375 by moving in an axial direction 1110 relative to the tubular member 375. The interface between the outer surface 1115 of the tapered conical portion 1120 of the expansion cone 370 and the  
15 inner surface 1125 of the tubular member 375 includes a leading edge portion 1130 and a trailing edge portion 1135.

During the radial expansion process, the leading and trailing edge portions, 1130 and 1135, are preferably lubricated by the presence of the coating 605 of lubricant. In a preferred embodiment, during the radial expansion process, the  
20 leading edge portion 5025 is further lubricated by the presence of lubricating fluids provided ahead of the expansion cone 370. However, because the radial clearance between the expansion cone 370 and the tubular member 375 in the trailing edge portion 1135 during the radial expansion process is typically extremely small, and the operating contact pressures between the tubular member 375 and the expansion  
25 cone 370 are extremely high, the quantity of lubricating fluid provided to the trailing edge portion 1135 is typically greatly reduced. In typical radial expansion operations, this reduction in the flow of lubricating fluids in the trailing edge portion 1135 increases the forces required to radially expand the tubular member 375.

30 Referring to FIG. 12, in a preferred embodiment, an expansion cone 1200 is used in the repair apparatus 300 that includes a front end 1200a, a rear end 1200b,

a tapered portion 1205 having an outer surface 1210, one or more circumferential grooves 1215a and 1215b, and one more internal flow passages 1220a and 1220b.

In a preferred embodiment, the circumferential grooves 1215 are fluidically coupled to the internal flow passages 1220. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1200a of the expansion cone 1200 into the circumferential grooves 1215. Thus, the trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. In a preferred embodiment, the lubricating fluids are injected into the internal flow passages 1220 using a fluid conduit that is coupled to the tapered end 1205 of the expansion cone 1200. Alternatively, lubricating fluids are provided for the internal flow passages 1220 using a supply of lubricating fluids provided adjacent to the front 1200a of the expansion cone 1200.

In a preferred embodiment, the expansion cone 1200 includes a plurality of circumferential grooves 1215. In a preferred embodiment, the cross sectional area of the circumferential grooves 1215 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the expansion cone 1200 includes circumferential grooves 1215 concentrated about the axial midpoint of the tapered portion 1205 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1200 and a tubular member during the radial expansion process. In a preferred embodiment, the circumferential grooves 1215 are equally spaced along the trailing edge portion of the expansion cone 1200 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1200 includes a plurality of flow passages 1220 coupled to each of the circumferential grooves 1215. In a preferred embodiment, the cross-sectional area of the flow passages 1220 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the



trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the cross sectional area of the circumferential grooves 1215 is greater than the cross sectional area of the flow passage 1220 in order to minimize  
5 resistance to fluid flow.

Referring to FIG. 13, in an alternative embodiment, an expansion cone 1300 is used in the repair apparatus 300 that includes a front end 1300a and a rear end 1300b, includes a tapered portion 1305 having an outer surface 1310, one or more circumferential grooves 1315a and 1315b, and one or more axial grooves 1320a and  
10 1320b.

In a preferred embodiment, the circumferential grooves 1315 are fluidically coupled to the axial grooves 1320. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1300a of the expansion cone 1300 into the circumferential grooves 1315. Thus, the trailing  
15 edge portion of the interface between the expansion cone 1300 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. In a preferred embodiment, the axial grooves 1320 are provided with lubricating fluid using a supply of lubricating fluid positioned proximate the front end 1300a of the  
20 expansion cone 1300. In a preferred embodiment, the circumferential grooves 1315 are concentrated about the axial midpoint of the tapered portion 1305 of the expansion cone 1300 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the  
25 circumferential grooves 1315 are equally spaced along the trailing edge portion of the expansion cone 1300 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1300 includes a plurality of  
30 circumferential grooves 1315. In a preferred embodiment, the cross sectional area of the circumferential grooves 1315 range from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-2}$  in<sup>2</sup> in order to optimally provide lubrication to the trailing edge portion of the interface

between the expansion cone 1300 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1300 includes a plurality of axial grooves 1320 coupled to each of the circumferential grooves 1315. In a preferred embodiment, the cross sectional area of the axial grooves 1320 ranges from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-2}$  in<sup>2</sup> in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the cross sectional area of the circumferential grooves 1315 is greater than the cross sectional area of the axial grooves 1320 in order to minimize resistance to fluid flow. In a preferred embodiment, the axial grooves 1320 are spaced apart in the circumferential direction by at least about 3 inches in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 14, in an alternative embodiment, an expansion cone 1400 is used in the repair apparatus 300 that includes a front end 1400a and a rear end 1400b, includes a tapered portion 1405 having an outer surface 1410, one or more circumferential grooves 1415a and 1415b, and one or more internal flow passages 1420a and 1420b.

In a preferred embodiment, the circumferential grooves 1415 are fluidically coupled to the internal flow passages 1420. In this manner, during the radial expansion process, lubricating fluids are transmitted from the areas in front of the front 1400a and/or behind the rear 1400b of the expansion cone 1400 into the circumferential grooves 1415. Thus, the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. Furthermore, the lubricating fluids also preferably pass to the area in front of the expansion cone 1400. In this manner, the area adjacent to the front 1400a of the expansion cone 1400 is cleaned of foreign materials. In a preferred embodiment, the lubricating fluids are injected into the internal flow passages 1420 by pressurizing the area behind the rear 1400b of the expansion cone 1400 during the radial expansion process.

In a preferred embodiment, the expansion cone 1400 includes a plurality of circumferential grooves 1415. In a preferred embodiment, the cross sectional area of the circumferential grooves 1415 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  respectively, in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the expansion cone 1400 includes circumferential grooves 1415 that are concentrated about the axial midpoint of the tapered portion 1405 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the circumferential grooves 1415 are equally spaced along the trailing edge portion of the expansion cone 1400 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1400 includes a plurality of flow passages 1420 coupled to each of the circumferential grooves 1415. In a preferred embodiment, the flow passages 1420 fluidically couple the front end 1400a and the rear end 1400b of the expansion cone 1400. In a preferred embodiment, the cross-sectional area of the flow passages 1420 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the cross sectional area of the circumferential grooves 1415 is greater than the cross-sectional area of the flow passages 1420 in order to minimize resistance to fluid flow.

Referring to FIG. 15, an alternative embodiment of an expansion cone 1500 is used in the apparatus that includes a front end 1500a and a rear end 1500b, includes a tapered portion 1505 having an outer surface 1510, one or more circumferential grooves 1515a and 1515b, and one or more axial grooves 1520a and 1520b.

In a preferred embodiment, the circumferential grooves 1515 are fluidically coupled to the axial grooves 1520. In this manner, during the radial expansion process, lubricating fluids are transmitted from the areas in front of the front 1500a

and/or behind the rear 1500b of the expansion cone 1500 into the circumferential grooves 1515. Thus, the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. Furthermore, in a preferred embodiment, pressurized lubricating fluids pass from the fluid passages 1520 to the area in front of the front 1500a of the expansion cone 1500. In this manner, the area adjacent to the front 1500a of the expansion cone 1500 is cleaned of foreign materials. In a preferred embodiment, the lubricating fluids are injected into the internal flow passages 1520 by pressurizing the area behind the rear 1500b expansion cone 1500 during the radial expansion process.

In a preferred embodiment, the expansion cone 1500 includes a plurality of circumferential grooves 1515. In a preferred embodiment, the cross sectional area of the circumferential grooves 1515 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the expansion cone 1500 includes circumferential grooves 1515 that are concentrated about the axial midpoint of the tapered portion 1505 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the circumferential grooves 1515 are equally spaced along the trailing edge portion of the expansion cone 1500 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1500 includes a plurality of axial grooves 1520 coupled to each of the circumferential grooves 1515. In a preferred embodiment, the axial grooves 1520 fluidically couple the front end and the rear end of the expansion cone 1500. In a preferred embodiment, the cross sectional area of the axial grooves 1520 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$ , respectively, in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial

expansion process. In a preferred embodiment, the cross sectional area of the circumferential grooves 1515 is greater than the cross sectional area of the axial grooves 1520 in order to minimize resistance to fluid flow. In a preferred embodiment, the axial grooves 1520 are spaced apart in the circumferential direction by at least about 3 inches in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 16, in an alternative embodiment, an expansion cone 1600 is used in the repair apparatus 300 that includes a front end 1600a and a rear end 1600b, includes a tapered portion 1605 having an outer surface 1610, one or more circumferential grooves 1615a and 1615b, and one or more axial grooves 1620a and 1620b.

In a preferred embodiment, the circumferential grooves 1615 are fluidically coupled to the axial grooves 1620. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1600a of the expansion cone 1600 into the circumferential grooves 1615. Thus, the trailing edge portion of the interface between the expansion cone 1600 and a tubular member is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. In a preferred embodiment, the lubricating fluids are injected into the axial grooves 1620 using a fluid conduit that is coupled to the tapered end 3205 of the expansion cone 1600.

In a preferred embodiment, the expansion cone 1600 includes a plurality of circumferential grooves 1615. In a preferred embodiment, the cross sectional area of the circumferential grooves 1615 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the expansion cone 1600 includes circumferential grooves 1615 that are concentrated about the axial midpoint of the tapered portion 1605 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the circumferential grooves 1615 are equally spaced along the trailing edge portion of the expansion cone 1600 in order to optimally provide lubrication to the trailing

edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1600 includes a plurality of axial grooves 1620 coupled to each of the circumferential grooves 1615. In a preferred embodiment, the axial grooves 1620 intersect each of the circumferential grooves 1615 at an acute angle. In a preferred embodiment, the cross sectional area of the axial grooves 1620 ranges from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-3}$  in<sup>2</sup> in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the cross sectional area of the circumferential grooves 1615 is greater than the cross sectional area of the axial grooves 1620. In a preferred embodiment, the axial grooves 1620 are spaced apart in the circumferential direction by at least about 3 inches in order to optimally provide lubrication during the radial expansion process. In a preferred embodiment, the axial grooves 1620 intersect the longitudinal axis of the expansion cone 1600 at a larger angle than the angle of attack of the tapered portion 1605 in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 17, in an alternative embodiment, an expansion cone 1700 is used in the repair apparatus 300 that includes a front end 1700a and a rear end 1700b, includes a tapered portion 1705 having an outer surface 1710, a spiral circumferential groove 1715, and one or more internal flow passages 1720.

In a preferred embodiment, the circumferential groove 1715 is fluidically coupled to the internal flow passage 1720. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1700a of the expansion cone 1700 into the circumferential groove 1715. Thus, the trailing edge portion of the interface between the expansion cone 1700 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member. In a preferred embodiment, the lubricating fluids are injected into the internal flow passage 1720 using a fluid conduit that is coupled to the tapered end 1705 of the expansion cone 1700.

In a preferred embodiment, the expansion cone 1700 includes a plurality of spiral circumferential grooves 1715. In a preferred embodiment, the cross sectional area of the circumferential groove 1715 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface  
5 between the expansion cone 1700 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the expansion cone 1700 includes circumferential grooves 1715 that are concentrated about the axial midpoint of the tapered portion 1705 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the tubular member  
10 370 during the radial expansion process. In a preferred embodiment, the circumferential grooves 1715 are equally spaced along the trailing edge portion of the expansion cone 1700 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the tubular member 370 during the radial expansion process.

15 In a preferred embodiment, the expansion cone 1700 includes a plurality of flow passages 1720 coupled to each of the circumferential grooves 1715. In a preferred embodiment, the cross-sectional area of the flow passages 1720 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the  
20 tubular member 370 during the radial expansion process. In a preferred embodiment, the cross sectional area of the circumferential groove 1715 is greater than the cross sectional area of the flow passage 1720 in order to minimize resistance to fluid flow.

Referring to FIG. 18, in an alternative embodiment, an expansion cone 1800  
25 is used in the repair apparatus 300 that includes a front end 1800a and a rear end 1800b, includes a tapered portion 1805 having an outer surface 1810, a spiral circumferential groove 1815, and one or more axial grooves 1820a, 1820b and 1820c.

In a preferred embodiment, the circumferential groove 1815 is fluidically coupled to the axial grooves 1820. In this manner, during the radial expansion  
30 process, lubricating fluids are transmitted from the area ahead of the front 1800a of the expansion cone 1800 into the circumferential groove 1815. Thus, the trailing edge portion of the interface between the expansion cone 1800 and a tubular

member is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. In a preferred embodiment, the lubricating fluids are injected into the axial grooves 1820 using a fluid conduit that is coupled to the tapered end 1805 of the expansion cone 1800.

5 In a preferred embodiment, the expansion cone 1800 includes a plurality of spiral circumferential grooves 1815. In a preferred embodiment, the cross sectional area of the circumferential grooves 1815 range from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-2}$  in<sup>2</sup> in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial  
10 expansion process. In a preferred embodiment, the expansion cone 1800 includes circumferential grooves 1815 concentrated about the axial midpoint of the tapered portion 1805 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the  
15 circumferential grooves 1815 are equally spaced along the trailing edge portion of the expansion cone 1800 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1800 includes a plurality of  
20 axial grooves 1820 coupled to each of the circumferential grooves 1815. In a preferred embodiment, the cross sectional area of the axial grooves 1820 range from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-2}$  in<sup>2</sup> in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the  
25 axial grooves 1820 intersect the circumferential grooves 1815 in a perpendicular manner. In a preferred embodiment, the cross sectional area of the circumferential groove 1815 is greater than the cross sectional area of the axial grooves 1820 in order to minimize resistance to fluid flow. In a preferred embodiment, the circumferential spacing of the axial grooves is greater than about 3 inches in order  
30 to optimally provide lubrication during the radial expansion process. In a preferred embodiment, the axial grooves 1820 intersect the longitudinal axis of the expansion



cone at an angle greater than the angle of attack of the tapered portion 1805 in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 19, in an alternative embodiment, an expansion cone 1900 is used in the repair apparatus 300 that includes a front end 1900a and a rear end 5 1900b, includes a tapered portion 1905 having an outer surface 1910, a circumferential groove 1915, a first axial groove 1920, and one or more second axial grooves 1925a, 1925b, 1925c and 1925d.

In a preferred embodiment, the circumferential groove 1915 is fluidically coupled to the axial grooves 1920 and 1925. In this manner, during the radial 10 expansion process, lubricating fluids are preferably transmitted from the area behind the back 1900b of the expansion cone 1900 into the circumferential groove 1915. Thus, the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular 15 member 370. In a preferred embodiment, the lubricating fluids are injected into the first axial groove 1920 by pressurizing the region behind the back 1900b of the expansion cone 1900. In a preferred embodiment, the lubricant is further transmitted into the second axial grooves 1925 where the lubricant preferably cleans foreign materials from the tapered portion 1905 of the expansion cone 1900.

20 In a preferred embodiment, the expansion cone 1900 includes a plurality of circumferential grooves 1915. In a preferred embodiment, the cross sectional area of the circumferential groove 1915 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial 25 expansion process. In a preferred embodiment, the expansion cone 1900 includes circumferential grooves 1915 concentrated about the axial midpoint of the tapered portion 1905 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the 30 circumferential grooves 1915 are equally spaced along the trailing edge portion of the expansion cone 1900 in order to optimally provide lubrication to the trailing

edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process.

In a preferred embodiment, the expansion cone 1900 includes a plurality of first axial grooves 1920 coupled to each of the circumferential grooves 1915. In a preferred embodiment, the first axial grooves 1920 extend from the back 1900b of the expansion cone 1900 and intersect the circumferential groove 1915. In a preferred embodiment, the cross sectional area of the first axial groove 1920 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the first axial groove 1920 intersects the circumferential groove 1915 in a perpendicular manner. In a preferred embodiment, the cross sectional area of the circumferential groove 1915 is greater than the cross sectional area of the first axial groove 1920 in order to minimize resistance to fluid flow. In a preferred embodiment, the circumferential spacing of the first axial grooves 1920 is greater than about 3 inches in order to optimally provide lubrication during the radial expansion process.

In a preferred embodiment, the expansion cone 1900 includes a plurality of second axial grooves 1925 coupled to each of the circumferential grooves 1915. In a preferred embodiment, the second axial grooves 1925 extend from the front 1900a of the expansion cone 1900 and intersect the circumferential groove 1915. In a preferred embodiment, the cross sectional area of the second axial grooves 1925 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. In a preferred embodiment, the second axial grooves 1925 intersect the circumferential groove 1915 in a perpendicular manner. In a preferred embodiment, the cross sectional area of the circumferential groove 1915 is greater than the cross sectional area of the second axial grooves 1925 in order to minimize resistance to fluid flow. In a preferred embodiment, the circumferential spacing of the second axial grooves 1925 is greater than about 3 inches in order to optimally provide lubrication during the radial expansion process. In a preferred embodiment, the second axial grooves 1925

intersect the longitudinal axis of the expansion cone 1900 at an angle greater than the angle of attack of the tapered portion 1905 in order to optimally provide lubrication during the radial expansion process.

Referring to Fig. 20, in a preferred embodiment, the first axial groove 1920 includes a first portion 2005 having a first radius of curvature 2010, a second portion 2015 having a second radius of curvature 2020, and a third portion 2025 having a third radius of curvature 2030. In a preferred embodiment, the radius of curvatures, 2010, 2020 and 2030 are substantially equal. In an exemplary embodiment, the radius of curvatures, 2010, 2020 and 2030 are all substantially equal to 0.0625 inches.

Referring to Fig. 21, in a preferred embodiment, the circumferential groove 1915 includes a first portion 2105 having a first radius of curvature 2110, a second portion 2115 having a second radius of curvature 2120, and a third portion 2125 having a third radius of curvature 2130. In a preferred embodiment, the radius of curvatures, 2110, 2120 and 2130 are substantially equal. In an exemplary embodiment, the radius of curvatures, 2110, 2120 and 2130 are all substantially equal to 0.125 inches.

Referring to Fig. 22, in a preferred embodiment, the second axial groove 1925 includes a first portion 2205 having a first radius of curvature 2210, a second portion 2215 having a second radius of curvature 2220, and a third portion 2225 having a third radius of curvature 2230. In a preferred embodiment, the first radius of curvature 2210 is greater than the third radius of curvature 2230. In an exemplary embodiment, the first radius of curvature 2210 is equal to 0.5 inches, the second radius of curvature 2220 is equal to 0.0625 inches, and the third radius of curvature 2230 is equal to 0.125 inches.

Referring to Fig. 23, in an alternative embodiment, an expansion cone 2300 is used in the repair apparatus 300 that includes an internal flow passage 2305 having an insert 2310 including a flow passage 2315. In a preferred embodiment, the cross sectional area of the flow passage 2315 is less than the cross sectional area of the flow passage 2305. More generally, in a preferred embodiment, a plurality of inserts 2310 are provided, each with different sizes of flow passages 2315. In this manner, the flow passage 2305 is machined to a standard size, and the lubricant

supply is varied by using different sized inserts 2310. In a preferred embodiment, the teachings of the expansion cone 2300 are incorporated into the expansion cones 1200, 1300, 1400, and 1700.

Referring to Fig. 24, in a preferred embodiment, the insert 2310 includes a filter 2405 for filtering particles and other foreign materials from the lubricant that passes into the flow passage 2305. In this manner, the foreign materials are prevented from clogging the flow passage 2305 and other flow passages within the expansion cone 2300.

The increased lubrication provided to the trailing edge portion of the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800, and 1900 greatly reduces the amount of galling or seizure caused by the interface between the expansion cones and the tubular member 370 during the radial expansion process thereby permitting larger continuous sections of tubulars to be radially expanded in a single continuous operation. Thus, use of the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800, and 1900 reduces the operating pressures required for radial expansion and thereby reduces the size of the pump 325. In addition, failure, bursting, and/or buckling of the tubular member 370 during the radial expansion process is significantly reduced, and the success ratio of the radial expansion process is greatly increased.

In a preferred embodiment, the lubricating fluids used with the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800 and 1900 for expanding the tubular member 370 have viscosities ranging from about 1 to 10,000 centipoise in order to optimize the injection of the lubricating fluids into the circumferential grooves of the expansion cones during the radial expansion process. In a preferred embodiment, the lubricating fluids used with the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800 and 1900 for expanding the tubular member 370 comprise various conventional lubricants available from various commercial vendors consistent with the teachings of the present disclosure in order to optimize the injection of the lubricating fluids into the circumferential grooves of the expansion cones during the radial expansion process.

In a preferred embodiment, as illustrated in FIG. 25, the expansion cone 375 further includes a central passage 2505 for receiving the support member 340 and

the repair apparatus 300 further includes one or more sealing members 2510 and one or more bearing members 2515.

The sealing members 2510 are preferably adapted to fluidically seal the dynamic interface between the central passage 2505 of the expansion cone 375 and the support member 340. The sealing members 2510 may be any number of conventional commercially available sealing members. In a preferred embodiment, the sealing members 2510 are conventional O-rings sealing members available from various commercial suppliers in order to optimally provide a fluidic seal.

The bearing members 2515 are preferably adapted to provide a sliding interface between the central passage 2505 of the expansion cone 375 and the support member 340. The bearing members 2515 may be any number of conventional commercially available bearings. In a preferred embodiment, the bearing members 2515 are wear bands available from Haliburton Energy Services in order to optimally provide a sliding interface that minimizes wear.

The sealing member 380 is coupled to the exterior surface of the expandable tubular member 375. The sealing member 380 is preferably adapted to fluidically seal the interface between the expandable tubular member 375 and the wellbore casing 100 after the radial expansion of the expandable tubular member 375. The sealing member 380 may be any number of conventional commercially available sealing members. In a preferred embodiment, the sealing member 380 is a nitrile rubber sealing member available from Eustler, Inc. in order to optimally provide a high pressure, high load bearing seal between the expandable tubular member 375 and the casing 100.

As illustrated in FIG. 3a, in a preferred embodiment, during placement of the repair apparatus 300 within the wellbore casing 100, the repair apparatus 300 is supported by the support member 305. In a preferred embodiment, during placement of the repair apparatus 300 within the wellbore casing 100, fluidic materials within the wellbore casing 100 are conveyed to a location above the repair apparatus 300 using the fluid conduits 335, 345, and 355. In this manner, surge pressures during placement of the repair apparatus 300 within the wellbore casing 100 are minimized.

In a preferred embodiment, prior to placement of the repair apparatus 300 in the wellbore, the outer surfaces of the repair apparatus 300 are coated with a lubricating fluid to facilitate their placement the wellbore and reduce surge pressures. In a preferred embodiment, the lubricating fluid comprises BARO-LUB  
5 GOLD-SEAL™ brand drilling mud lubricant, available from Baroid Drilling Fluids, Inc. In this manner, the insertion of the repair apparatus 300 into the wellbore casing 100 is optimized.

In a preferred embodiment, after placement of the repair apparatus 300 within the wellbore casing 100, in step 210, the logging tool 310 is used in a  
10 conventional manner to locate the openings 115 in the wellbore casing 100.

In a preferred embodiment, once the openings 115 have been located by the logging tool 310, in step 215, the repair apparatus 300 is further positioned within the wellbore casing 100 with the sealing member 380 placed in opposition to the openings 115.

As illustrated in FIGS. 3b and 3c, in a preferred embodiment, after the repair  
15 apparatus 300 has been positioned with the sealing member 380 in opposition to the openings 115, in step 220, the tubular member 370 is radially expanded into contact with the wellbore casing 100. In a preferred embodiment, the tubular member 370 is radially expanded by displacing the expansion cone 375 in the axial direction. In  
20 a preferred embodiment, the expansion cone 375 is displaced in the axial direction by pressurizing the interior portion 385. In a preferred embodiment, the interior portion 385 is pressurized by pumping fluidic materials into the interior portion 385 using the pump 325.

In a preferred embodiment, the pump 325 pumps fluidic materials from the  
25 region above and proximate to the repair apparatus 300 into the interior portion 385 using the fluidic passages 320 and 330. In this manner, the interior portion 385 is pressurized and the expansion cone 375 is displaced in the axial direction. In this manner, the tubular member 370 is radially expanded into contact with the wellbore casing 100. In a preferred embodiment, the interior portion 385 is pressurized to  
30 operating pressures ranging from about 0 to 12,000 psi using flow rates ranging from about 0 to 500 gallons/minute. In a preferred embodiment, fluidic materials displaced by the axial movement of the expansion cone 375 are conveyed to a

location above the repair apparatus 300 by the fluid conduits 335, 345, and 355. In a preferred embodiment, during the pumping of fluidic materials into the interior portion 385 by the pump 325, the tubular member 370 is maintained in a substantially stationary position.

5       As illustrated in FIG. 3d, after the completion of the radial expansion of the tubular member 370, the locking member 365 is decoupled from the tubular member 370 and the repair apparatus 300 is removed from the wellbore casing 100. In a preferred embodiment, during the removal of the repair apparatus 300 from the wellbore casing 100, fluidic materials above the repair apparatus 300 are conveyed  
10 to a location below the repair apparatus 300 using the fluid conduits 335, 345 and 355. In this manner, the removal of the repair apparatus 300 from the wellbore casing is facilitated.

As illustrated in FIG. 3e, in a preferred embodiment, the openings 115 in the wellbore casing 100 are sealed off by the radially expanded tubular member 370 and  
15 the sealing member 380. In this manner, the repair apparatus 300 provides a compact and efficient device for repairing wellbore casings. More generally, the repair apparatus 300 is used to repair and form wellbore casings, pipelines, and structural supports.

Referring to FIG. 26a, in an alternative embodiment, in step 205, a repair  
20 apparatus 2600 is positioned within the wellbore casing 100.

The repair apparatus 2600 preferably includes a first support member 2605, a logging tool 2610, a housing 2615, a first fluid conduit 2620, a pump 2625, a second fluid conduit 2630, a first valve 2635, a third fluid conduit 2640, a second valve 2645, a fourth fluid conduit 2650, a second support member 2655, a fifth fluid  
25 conduit 2660, the third support member 2665, a sixth fluid conduit 2670, sealing members 2675, a locking member 2680, an expandable tubular 2685, an expansion cone 2690, a sealing member 2695, a packer 2700, a seventh fluid conduit 2705, and a third valve 2710.

The first support member 2605 is preferably coupled to the logging tool 2610  
30 and the housing 2615. The first support member 2605 is preferably adapted to be coupled to and supported by a conventional support member such as, for example, a wireline or a drill string. The first support member 2605 preferably has a

substantially annular cross section in order to provide one or more conduits for conveying fluidic materials from the apparatus 2600. The first support member 2605 is further preferably adapted to convey electrical power and communication signals to the logging tool 2610, the pump 2625, the valves 2635, 2645, and 2710, 5 and the packer 2700.

The logging tool 2610 is preferably coupled to the first support member 2605. The logging tool 2610 is preferably adapted to detect defects in the wellbore casing 100. The logging tool 2610 may be any number of conventional commercially available logging tools suitable for detecting defects in wellbore casings, pipelines, 10 or structural supports. In a preferred embodiment, the logging tool 2610 is a CAST logging tool, available from Halliburton Energy Services in order to optimally provide detection of defects in the wellbore casing 100. In a preferred embodiment, the logging tool 2610 is contained within the housing 2615 in order to provide a repair apparatus 2600 that is rugged and compact.

15 The housing 2615 is preferably coupled to the first support member 2605, the second support member 2655, the sealing members 2675, and the locking member 2680. The housing 2615 is preferably releasably coupled to the tubular member 2685. The housing 2615 is further preferably adapted to contain and support the logging tool 2610 and the pump 2625.

20 The first fluid conduit 2620 is preferably fluidically coupled to the inlet of the pump 2625, the exterior region above the housing 2615, and the second fluid conduit 2630. The first fluid conduit 2620 may be contained within the first support member 2605 and the housing 2615. The first fluid conduit 2620 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and 25 lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 2690.

The pump 2625 is fluidically coupled to the first fluid conduit 2620 and the third fluid conduit 2640. The pump 2625 is further preferably contained within and 30 support by the housing 2615. The pump 2625 is preferably adapted to convey fluidic materials from the first fluid conduit 2620 to the third fluid conduit 2640 at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500



gallons/minute in order to optimally provide operating pressure for propagating the expansion cone 2690. The pump 2625 may be any number of conventional commercially available pumps. In a preferred embodiment, the pump 2625 is a flow control pump out section, available from Halliburton Energy Services in order to  
5 optimally provide fluid pressure for propagating the expansion cone 2690. The pump 2625 is preferably adapted to pressurize an interior portion 2715 of the expandable tubular member 2685 to operating pressures ranging from about 0 to 12,000 psi.

The second fluid conduit 2630 is fluidically coupled to the first fluid conduit  
10 2620 and the third fluid conduit 2640. The second fluid conduit 2630 is further preferably contained within the housing 2615. The second fluid conduit 2630 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally provide propagation  
15 of the expansion cone 2690.

The first valve 2635 is preferably adapted to controllably block the second fluid conduit 2630. In this manner, the flow of fluidic materials through the second fluid conduit 2630 is controlled. The first valve 2635 may be any number of conventional commercially available flow control valves. In a preferred  
20 embodiment, the first valve 2635 is a conventional ball valve available from various commercial suppliers.

The third fluid conduit 2640 is fluidically coupled to the outlet of the pump 2625, the second fluid conduit 2630, and the fifth fluid conduit 2660. The third fluid conduit 2640 is further preferably contained within the housing 2615. The third  
25 fluid conduit 2640 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally provide propagation of the expansion cone 2690.

The second valve 2645 is preferably adapted to controllably block the third  
30 fluid conduit 2640. In this manner, the flow of fluidic materials through the third fluid conduit 2640 is controlled. The second valve 2645 may be any number of conventional commercially available flow control valves. In a preferred

embodiment, the second valve 2645 is a conventional ball valve available from various commercial sources.

The fourth fluid conduit 2650 is fluidically coupled to the exterior region above the housing 2615 and the interior region 2720 within the expandable tubular member 2685. The fourth fluid conduit 2650 is further preferably contained within the housing 2615. The fourth fluid conduit 2650 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 5,000 psi and 0 to 500 gallons/minute in order to optimally vent fluidic materials in front of the expansion cone 2690 during the radial expansion process.

The second support member 2655 is coupled to the housing 2615 and the third support member 2665. The second support member 2655 is further preferably movably and sealingly coupled to the expansion cone 2690. The second support member 2655 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials. In a preferred embodiment, the second support member 2655 is centrally positioned within the expandable tubular member 2685.

The fifth fluid conduit 2660 is fluidically coupled to the third fluid conduit 2640 and the sixth fluid conduit 2670. The fifth fluid conduit 2660 is further preferably contained within the second support member 2655. The fifth fluid conduit 2660 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 2690.

The third support member 2665 is coupled to the second support member 2655. The third support member 2665 is further preferably adapted to support the expansion cone 2690. The third support member 2665 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials.

The sixth fluid conduit 2670 is fluidically coupled to the fifth fluid conduit 2660 and the interior region 2715 of the expandable tubular member 2685 below the expansion cone 2690. The sixth fluid conduit 2670 is further preferably contained

within the third support member 2665. The sixth fluid conduit 2670 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone  
5 2690.

The sealing members 2675 are preferably coupled to the housing 2615. The sealing members 2675 are preferably adapted to seal the interface between the exterior surface of the housing 2615 and the interior surface of the expandable tubular member 2685. In this manner, the interior portion 2730 of the expandable  
10 tubular member 2685 is fluidically isolated from the exterior region above the housing 2615. The sealing members 2675 may be any number of conventional commercially available sealing members. In a preferred embodiment, the sealing members 2675 are conventional O-ring sealing members available from various commercial suppliers in order to optimally provide a pressure seal.

15 The locking member 2680 is preferably coupled to the housing 2615. The locking member 2680 is further preferably releasably coupled to the expandable tubular member 2685. In this manner, the housing 2615 is controllably coupled to the expandable tubular member 2685. In this manner, the housing 2615 is preferably released from the expandable tubular member 2685 upon the completion  
20 of the radial expansion of the expandable tubular member 2685. The locking member 2680 may be any number of conventional commercially available releasable locking members. In a preferred embodiment, the locking member 2680 is a hydraulically released slip available from various commercial vendors in order to optimally provide support during the radial expansion process.

25 In an alternative embodiment, the locking member 2680 is replaced by or supplemented by one or more conventional shear pins in order to provide an alternative means of controllably releasing the housing 2615 from the expandable tubular member 2685.

In another alternative embodiment, the seals 2675 and locking member 2680  
30 are omitted.

The expandable tubular member 2685 is releasably coupled to the locking member 2680. The expandable tubular member 2685 is preferably adapted to be

radially expanded by the axial displacement of the expansion cone 2690. In a preferred embodiment, the expandable tubular member 2685 is substantially identical to the expandable tubular member 370 described above with reference to the repair apparatus 300.

5       The expansion cone 2690 is movably coupled to the second support member 2655. The expansion cone 2690 is preferably adapted to be axially displaced upon the pressurization of the interior region 2715 of the expandable tubular member 2685. The expansion cone 2690 is further preferably adapted to radially expand the expandable tubular member 2685. In a preferred embodiment, the expansion cone  
10 2690 is substantially identical to the expansion cone 375 described above with reference to the repair apparatus 300.

      The sealing member 2695 is coupled to the exterior surface of the expandable tubular member 2685. The sealing member 2695 is preferably adapted to fluidically seal the interface between the expandable tubular member 2685 and the wellbore  
15 casing 100 after the radial expansion of the expandable tubular member 2685. The sealing member 2695 may be any number of conventional commercially available sealing members. In a preferred embodiment, the sealing member 2695 is a nitrile rubber sealing member available from Eustler, Inc. in order to optimally provide a high pressure seal between the casing 100 and the expandable tubular member  
20 2685.

      The packer 2700 is coupled to the third support member 2665. The packer 2700 is further releasably coupled to the expandable tubular member 2685. The packer 2700 is preferably adapted to fluidically seal the interior region 2715 of the expandable tubular member 2685. In this manner, the interior region 2715 of the  
25 expandable tubular member 2685 is pressurized. The packer 2700 may be any number of conventional commercially available packer devices. In a preferred embodiment, the packer 2700 is an EZ Drill Packer available from Halliburton Energy Services in order to optimally provide a high pressure seal below the expansion cone 2690 that can be easily removed upon the completion of the radial  
30 expansion process.

      The seventh fluid conduit 2705 is fluidically coupled to the interior region 2715 of the expandable tubular member 2685 and an exterior region below the apparatus

2600. The seventh fluid conduit 2705 is further preferably contained within the packer 2700. The seventh fluid conduit 2705 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 1,500 psi and 0 to 200  
5 gallons/minute in order to optimally provide a fluid conduit that minimizes back pressure on the apparatus 2600 when the apparatus 2600 is positioned within the wellbore casing 100.

The third valve 2710 is preferably adapted to controllably block the seventh fluid conduit 2705. In this manner, the flow of fluidic materials through the seventh  
10 fluid conduit 2705 is controlled. The third valve 2710 may be any number of conventional commercially available flow control valves. In a preferred embodiment, the third valve 2710 is a EZ Drill one-way check valve available from Halliburton Energy Services in order to optimally provide one-way flow through the packer 2700 while providing a pressure seal during the radial expansion process.

15 As illustrated in FIG. 26a, in a preferred embodiment, during placement of the repair apparatus 2600 within the wellbore casing 100, the apparatus 2600 is supported by the support member 2605. In a preferred embodiment, during placement of the apparatus 2600 within the wellbore casing 100, fluidic materials within the wellbore casing 100 are conveyed to a location above the apparatus 2600  
20 using the fluid conduits 2705, 2670, 2660, 2640, 2630, and 2620. In this manner, surge pressures during placement of the apparatus 2600 within the wellbore casing 100 are minimized.

In a preferred embodiment, prior to placement of the apparatus 2600 in the wellbore casing 100, the outer surfaces of the apparatus 2600 are coated with a  
25 lubricating fluid to facilitate their placement the wellbore and reduce surge pressures. In a preferred embodiment, the lubricating fluid comprises BARO-LUB GOLD-SEAL™ brand drilling mud lubricant, available from Baroid Drilling Fluids, Inc. In this manner, the insertion of the apparatus 2600 into the wellbore casing 100 is optimized.

30 In a preferred embodiment, after placement of the apparatus 2600 within the wellbore casing 100, in step 210, the logging tool 2610 is used in a conventional manner to locate the openings 115 in the wellbore casing 100.

In a preferred embodiment, once the openings 115 have been located by the logging tool 2610, in step 215, the apparatus 2600 is further positioned within the wellbore casing 100 with the sealing member 2695 placed in opposition to the openings 115.

5       As illustrated in FIGS. 26b and 26c, in a preferred embodiment, after the apparatus 2600 has been positioned with the sealing member 2695 in opposition to the openings 115, in step 220, the tubular member 2685 is radially expanded into contact with the wellbore casing 100. In a preferred embodiment, the tubular member 2685 is radially expanded by displacing the expansion cone 2690 in the axial  
10       direction. In a preferred embodiment, the expansion cone 2690 is displaced in the axial direction by pressurizing the interior chamber 2715. In a preferred embodiment, the interior chamber 2715 is pressurized by pumping fluidic materials into the interior chamber 2715 using the pump 2625.

In a preferred embodiment, the pump 2625 pumps fluidic materials from the  
15       region above and proximate to the apparatus 2600 into the interior chamber 2715 using the fluid conduits 2620, 2640, 2660, and 2670. In this manner, the interior chamber 2715 is pressurized and the expansion cone 2690 is displaced in the axial direction. In this manner, the tubular member 2685 is radially expanded into contact with the wellbore casing 100. In a preferred embodiment, the interior  
20       chamber 2715 is pressurized to operating pressures ranging from about 0 to 12,000 psi using flow rates ranging from about 0 to 500 gallons/minute. In a preferred embodiment, fluidic materials within the interior chamber 2720 displaced by the axial movement of the expansion cone 2690 are conveyed to a location above the apparatus 2600 by the fluid conduit 2650. In a preferred embodiment, during the  
25       pumping of fluidic materials into the interior chamber 2715 by the pump 2625, the tubular member 2685 is maintained in a substantially stationary position.

As illustrated in FIG. 26d, after the completion of the radial expansion of the tubular member 2685, the locking member 2680 and packer 2700 are decoupled from the tubular member 2685, and the apparatus 2600 is removed from the  
30       wellbore casing 100. In a preferred embodiment, during the removal of the apparatus 2600 from the wellbore casing 100, fluidic materials above the apparatus 2600 are conveyed to a location below the apparatus 2600 using the fluid conduits

2620, 2630, 2640, 2660, and 2670. In this manner, the removal of the apparatus 2600 from the wellbore casing is facilitated.

As illustrated in FIG. 26e, in a preferred embodiment, the openings 115 in the wellbore casing 100 are sealed off by the radially expanded tubular member 2685 and the sealing member 2695. In this manner, the repair apparatus 2600 provides a compact and efficient device for repairing wellbore casings. More generally, the repair apparatus 2600 is used to repair and form wellbore casings, pipelines, and structural supports.

A method of repairing an opening in a tubular member has been described that includes positioning an expandable tubular, an expansion cone, and a pump within the tubular member, positioning the expandable tubular in opposition to the opening in the tubular member, pressurizing an interior portion of the expandable tubular using the pump, and radially expanding the expandable tubular into intimate contact with the tubular member using the expansion cone. In a preferred embodiment, the method further includes locating the opening in the tubular member using an opening locator. In a preferred embodiment, the tubular member is a wellbore casing. In a preferred embodiment, the tubular member is a pipeline. In a preferred embodiment, the tubular member is a structural support. In a preferred embodiment, the method further includes lubricating the interface between the expandable tubular member and the expansion cone. In a preferred embodiment, lubricating includes coating the expandable tubular member with a lubricant. In a preferred embodiment, lubricating includes injecting a lubricating fluid into the trailing edge of the interface between the expandable tubular member and the expansion cone. In a preferred embodiment, lubricating includes coating the expandable tubular member with a first component of a lubricant and circulating a second component of the lubricant into contact with the coating on the expandable tubular member. In a preferred embodiment, the method further includes sealing off a portion of the expandable tubular member.

An apparatus for repairing a tubular member also has been described that includes a support member, an expandable tubular member removably coupled to the support member, an expansion cone movably coupled to the support member and a pump coupled to the support member adapted to pressurize a portion of the

interior of the expandable tubular member. In a preferred embodiment, the expandable tubular member includes a coating of a lubricant. In a preferred embodiment, the expandable tubular member includes a coating of a first component of a lubricant. In a preferred embodiment, the expandable tubular member includes a sealing member coupled to the outer surface of the expandable tubular member. In a preferred embodiment, the expandable tubular member includes a first end having a first outer diameter, an intermediate portion coupled to the first end having an intermediate outer diameter and a second end having a second outer diameter coupled to the intermediate portion having a second outer diameter, wherein the first and second outer diameters are greater than the intermediate outer diameter. In a preferred embodiment, the first end, second end, and intermediate portion of the expandable tubular member have wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  and inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ ; and the relationship between the wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$ , the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ , the inside diameter  $D_{TUBE}$  of the tubular member that the expandable tubular member will be inserted into, and the outside diameter  $D_{cone}$  of the expansion cone is given by the following expression:

$$D_{TUBE} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} \left[ (t_1 - t_{INT}) * D_{cone} + t_{INT} * D_{INT} \right]$$

where  $t_1 = t_2$ ; and  $D_1 = D_2$ . In a preferred embodiment, the expandable tubular member includes a sealing member coupled to the outside surface of the intermediate portion. In a preferred embodiment, the expandable tubular member includes a first transition portion coupled to the first end and the intermediate portion inclined at a first angle and a second transition portion coupled to the second end and the intermediate portion inclined at a second angle, wherein the first and second angles range from about 5 to 45 degrees. In a preferred embodiment, the expansion cone includes an expansion cone surface having an angle of attack ranging from about 10 to 40 degrees. In a preferred embodiment, the expansion cone includes a first expansion cone surface having a first angle of attack and a second expansion cone surface having a second angle of attack, wherein the first angle of attack is greater than the second angle of attack. In a preferred



embodiment, the expansion cone includes an expansion cone surface having a substantially parabolic profile. In a preferred embodiment, the expansion cone includes an inclined surface including one or more lubricating grooves. In a preferred embodiment, the expansion cone includes one or more internal lubricating  
5 passages coupled to each of the lubricating grooves.

A method of coupling a first tubular member to a second tubular member, wherein the outside diameter of the first tubular member is less than the inside diameter of the second tubular member also has been described that includes positioning at least a portion of the first tubular member within the second tubular  
10 member, pressurizing a portion of the interior of the first tubular member by pumping fluidic materials proximate the first tubular member into the portion of the interior of the first tubular member, and displacing an expansion cone within the interior of the first tubular member. In a preferred embodiment, the second tubular member is selected from the group  
15 consisting of a wellbore casing, a pipeline, and a structural support. In a preferred embodiment, the method further includes lubricating the interface between the first tubular member and the expansion cone. In a preferred embodiment, the lubricating includes coating the first tubular member with a lubricant. In a preferred embodiment, the lubricating includes injecting a lubricating fluid into the  
20 trailing edge of the interface between the first tubular member and the expansion cone. In a preferred embodiment, the lubricating includes coating the first tubular member with a first component of a lubricant and circulating a second component of the lubricant into contact with the coating on the first tubular member. In a preferred embodiment, the method further includes sealing off a portion of the first  
25 tubular member.

Although illustrative embodiments of the invention have been shown and described, a wide range of modification, changes and substitution is contemplated in the foregoing disclosure. In some instances, some features of the present invention may be employed without a corresponding use of the other features.  
30 Accordingly, it is appropriate that the appended claims be construed broadly and in a manner consistent with the scope of the invention.

- 47 -

Throughout this specification and the claims which follow, unless the context requires otherwise, the word "comprise", and variations such as "comprises" and "comprising", will be understood to imply the inclusion of a stated integer or step or group of integers or steps but not the exclusion of any other integer or step or group of integers or steps.

The reference to any prior art in this specification is not, and should not be taken as, an acknowledgment or any form of suggestion that that prior art forms part of the common general knowledge in Australia.



- 48 -

THE CLAIMS DEFINING THE INVENTION ARE AS FOLLOWS:

1. A method of repairing an opening in a tubular member, comprising:  
positioning an expandable tubular, an expansion cone, and a pump within the  
5 tubular member;  
positioning the expandable tubular in opposition to the opening in the tubular  
member;  
pressurising an interior portion of the expandable tubular by operating the pump  
within the tubular member; and  
10 radially expanding the expandable tubular into intimate contact with the tubular  
member using the expansion cone.
2. The method of claim 1, further comprising:  
locating the opening in the tubular member using an opening locator.
- 15 3. The method of claim 1 or 2, wherein the tubular member comprises a wellbore  
casing.
4. The method of claim 1 or 2, wherein the tubular member comprises a pipeline.
- 20 5. The method of claim 1 or 2, wherein the tubular member comprises a structural  
support.
6. The method of any one of claims 1 to 5, further comprising:  
25 lubricating the interface between the expandable tubular and the expansion cone.
7. The method of claim 6, wherein the lubricating comprises:  
coating the expandable tubular with lubricant.
- 30 8. The method of claim 6 or 7, wherein the lubricating comprises:

- 49 -

injecting a lubricating fluid into a trailing edge of the interface between the expandable tubular and the expansion cone.

9. The method of claim 6, wherein the lubricating comprises:  
5 coating the expandable tubular with a first component of a lubricant; and circulating a second component of the lubricant into contact with the coating on the expandable tubular.
10. The method of any one of the preceding claims, further comprising:  
10 sealing off a portion of the expandable tubular.
11. Apparatus for repairing a tubular member, comprising:  
a support member;  
an expandable tubular removably supported by the support member;  
15 an expansion cone movably supported by the support member; and a pump supported by the support member, the pump being positioned within the expandable tubular and being adapted to pressurise a portion of the interior of the expandable tubular .
- 20 12. The apparatus of claim 11, wherein the expandable tubular has a coating of lubricant.
13. The apparatus of claim 11, wherein the expandable tubular has a coating of a first component of a lubricant.  
25
14. The apparatus of any one of claims 11 to 13, wherein the expandable tubular has a sealing member on its outer surface.
15. The apparatus of any one of claims 11 to 14, wherein the expandable tubular  
30 comprises:  
a first end having a first outer diameter;

- 50 -

a second end having a second outer diameter; and  
 an intermediate portion between the first and second ends and having an  
 intermediate outer diameter;  
 wherein the first and second outer diameters are greater than the intermediate outer  
 5 diameter.

16. The apparatus of claim 15, wherein the first end, second end, and intermediate  
 portion of the expandable tubular have respective wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  and  
 respective inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ ; and wherein the relationship between the wall  
 10 thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$ , the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ , the inside diameter  $D_{TUBE}$   
 of the tubular member that the expandable tubular will be inserted into, and the outside  
 diameter  $D_{cone}$  of the expansion cone is given by the following expression:

$$D_{TUBE} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} [(t_1 - t_{INT}) * D_{cone} + t_{INT} * D_{INT}]$$

- 15 where  $t_1 = t_2$ ; and  
 $D_1 = D_2$ .

17. The apparatus of claim 15 or 16, when claim 15 is dependent from claim 14,  
 wherein the sealing member is on the outside surface of the intermediate portion.

18. The apparatus of any one of claims 15 to 17, wherein the expandable tubular  
 comprises:

a first transition portion between the first end and the intermediate portion inclined  
 at a first angle; and

25 a second transition portion between to the second end and the intermediate portion  
 inclined at a second angle;

wherein the first and second angles range from about 5 to 45 degrees.

19. The apparatus of any one of claims 11 to 18, wherein the expansion cone has an  
 30 expansion cone surface having an angle of attack ranging from about 10 to 40 degrees.

- 51 -

20. The apparatus of any one of claims 11 to 19, wherein the expansion cone comprises:

- a first expansion cone surface having a first angle of attack; and
- 5 a second expansion cone surface having a second angle of attack;
- wherein the first angle of attack is greater than the second angle of attack.

21. The apparatus of any one of claims 11 to 18, wherein the expansion cone has an expansion cone surface having a substantially parabolic profile.

10

22. The apparatus of any one of claims 11 to 21, wherein the expansion cone has an inclined surface including one or more lubricating grooves.

23. The apparatus of claim 22, wherein the expansion cone has one or more internal

15

24. A method of joining a first tubular member to a second tubular member, wherein the outside diameter of the first tubular member is less than the inside diameter of the second tubular member, comprising:

20

positioning at least a portion of the first tubular member within the second tubular member;

positioning a pump within the first tubular member;

pressurising a portion of the interior of the first tubular member by pumping fluidic materials proximate the first tubular member into the portion of the interior of the

25

first tubular member using the pump; and

displacing an expansion cone within the interior of the first tubular member to radially expand said at least a portion of the first tubular member into intimate contact with the second tubular member.

30

25. The method of claim 24, wherein the second tubular member is selected from a wellbore casing, a pipeline, and a structural support.

- 52 -

26. The method of claim 24 or 25, further comprising:  
lubricating the interface between the first tubular member and the expansion cone.
- 5 27. The method of claim 26, wherein the lubricating comprises:  
coating the first tubular member with a lubricant.
28. The method of claim 26 or 27, wherein the lubricating comprises:  
injecting a lubricating fluid into a trailing edge of the interface between the first  
10 tubular member and the expansion cone.
29. The method of claim 26, wherein the lubricating comprises:  
coating the first tubular member with a first component of a lubricant; and  
circulating a second component of the lubricant into contact with the coating on the  
15 first tubular member.
30. The method of any one of claims 24 to 29, further comprising sealing off a portion  
of the first tubular member.
- 20 31. A method of repairing an opening in a tubular member, comprising:  
positioning an expandable tubular, an expansion cone, and a pump within the  
tubular member;  
positioning the expandable tubular in opposition to the opening in the tubular  
member;  
25 injecting fluidic materials into an interior portion of the expandable tubular using  
the pump to pressurise the interior portion of the expandable tubular; and  
displacing the expansion cone relative to the expandable tubular to radially expand  
the expandable tubular into intimate contact with the tubular member.
- 30 32. The method of claim 31, further comprising:  
locating the opening in the tubular member using an opening locator.

- 53 -

33. The method of claim 31 or 32, wherein the tubular member comprises a wellbore casing.
- 5 34. The method of claim 31 or 32, wherein the tubular member comprises a pipeline.
35. The method of claim 31 or 32, wherein the tubular member comprises a structural support.
- 10 36. The method of any one of claims 31 to 35, further comprising:  
lubricating the interface between the expandable tubular and the expansion cone.
37. The method of claim 36, wherein the lubricating comprises:  
coating the expandable tubular with lubricant.
- 15 38. The method of claim 36 or 37, wherein the lubricating comprises:  
injecting a lubricating fluid into a trailing edge of the interface between the  
expandable tubular and the expansion cone.
- 20 39. The method of claim 36, wherein the lubricating comprises:  
coating the expandable tubular with a first component of a lubricant; and  
circulating a second component of the lubricant into contact with the coating on the  
expandable tubular.
- 25 40. The method of any one of claims 31 to 39, further comprising:  
sealing off a portion of the expandable tubular.
41. Apparatus for repairing a tubular member, comprising:  
a support member;  
30 an expandable tubular removably supported by the support member;  
a tubular expansion cone movably supported by the support member; and



- 54 -

a pump positioned within the expandable tubular supported by the support member and adapted to pressurise a portion of the interior of the expandable tubular.

42. The apparatus of claim 41, wherein the expandable tubular has a coating of  
5 lubricant.

43. The apparatus of claim 41, wherein the expandable tubular has a coating of a first component of a lubricant.

10 44. The apparatus of any one of claims 41 to 43, wherein the expandable tubular has a sealing member on its outer surface.

45. The apparatus of any one of claims 41 to 44, wherein the expandable tubular comprises:

15 a first end having a first outer diameter;  
a second end having a second outer diameter; and  
an intermediate portion between the first and second ends and having an intermediate outer diameter;  
wherein the first and second outer diameters are greater than the intermediate outer  
20 diameter.

46. The apparatus of claim 45, wherein the first end, second end, and intermediate  
portion of the expandable tubular have respective wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  and  
respective inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ ; and wherein the relationship between the wall  
25 thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$ , the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ , the inside diameter  $D_{TUBE}$   
of the tubular member that the expandable tubular will be inserted into, and the outside  
diameter  $D_{cone}$  of the expansion cone is given by the following expression:

$$D_{TUBE} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} [(t_1 - t_{INT}) * D_{cone} + t_{INT} * D_{INT}]$$

30 where  $t_1 = t_2$ ; and

- 55 -

$$D_1 = D_2.$$

47. The apparatus of claim 45 or 46, when claim 45 is dependent from claim 44, wherein the sealing member is on the outside surface of the intermediate portion.

5

48. The apparatus of any one of claims 45 to 47, wherein the expandable tubular comprises:

a first transition portion between the first end and the intermediate portion inclined at a first angle; and

10 a second transition portion between to the second end and the intermediate portion inclined at a second angle;

wherein the first and second angles range from about 5 to 45 degrees.

49. The apparatus of any one of claims 41 to 48, wherein the tubular expansion  
15 cone has an expansion cone surface having an angle of attack ranging from about 10 to 40 degrees.

50. The apparatus of any one of claims 41 to 49, wherein the tubular expansion cone comprises:

20 a first expansion cone surface having a first angle of attack; and  
a second expansion cone surface having a second angle of attack;  
wherein the first angle of attack is greater than the second angle of attack.

51. The apparatus of any one of claims 41 to 48, wherein the tubular expansion  
25 cone has an expansion cone surface having a substantially parabolic profile.

52. The apparatus of any one of claims 41 to 51, wherein the tubular expansion cone has an inclined surface including one or more lubricating grooves.

30 53. The apparatus of claim 52, wherein the expansion cone has one or more internal lubricating passages connected to each of the lubricating grooves.

- 56 -

54. A method of joining a first tubular member to a second tubular member, wherein the outside diameter of the first tubular member is less than the inside diameter of the second tubular member, comprising:

- 5       positioning at least a portion of the first tubular member within the second tubular member;
- positioning a pump within the first tubular member;
- pressurising a portion of the interior of the first tubular member by pumping fluidic materials proximate the first tubular member into the portion of the interior of the
- 10       first tubular member by operating the pump; and
- displacing a tubular expansion cone within the interior of the first tubular member to radially expand at least a portion of the first tubular member into intimate contact with the second tubular member.

- 15   55. The method of claim 54, wherein the second tubular member is selected from a wellbore casing, a pipeline, and a structural support.

56. The method of claim 54 or 55, further comprising:

20       lubricating the interface between the first tubular member and the tubular expansion cone.

57. The method of claim 56, wherein the lubricating comprises:

      coating the first tubular member with a lubricant.

25   58. The method of claim 56 or 57, wherein the lubricating comprises:

      injecting a lubricating fluid into a trailing edge of the interface between the first tubular member and the tubular expansion cone.

59. The method of claim 56, wherein the lubricating comprises:

30       coating the first tubular member with a first component of a lubricant; and

- 57 -

circulating a second component of the lubricant into contact with the coating on the first tubular member.

60. The method of any one of claims 54 to 59, further comprising sealing off a portion  
5 of the first tubular member.

61. A method of repairing an opening in a tubular member, substantially as herein described with reference to the accompanying drawings.

10 62. Apparatus for repairing a tubular member, substantially as herein described with reference to the accompanying drawings.

63. A method of joining two tubular members, substantially as herein described with reference to the accompanying drawings.

15

64. A tubular member which has been repaired by the method of any one of claims 1 to 10, 31 to 40 and 61 or by the apparatus of any one of claims 11 to 23, 41 to 55 and 62.

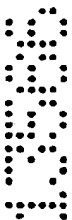
65. Tubular members which have been joined by the method of any one of claims 24 to  
20 30, 54 to 60 and 63.

Dated this 3rd day of June, 2005

**Shell Oil Company**

by their Patent Attorneys

25 DAVIES COLLISON CAVE



1/30

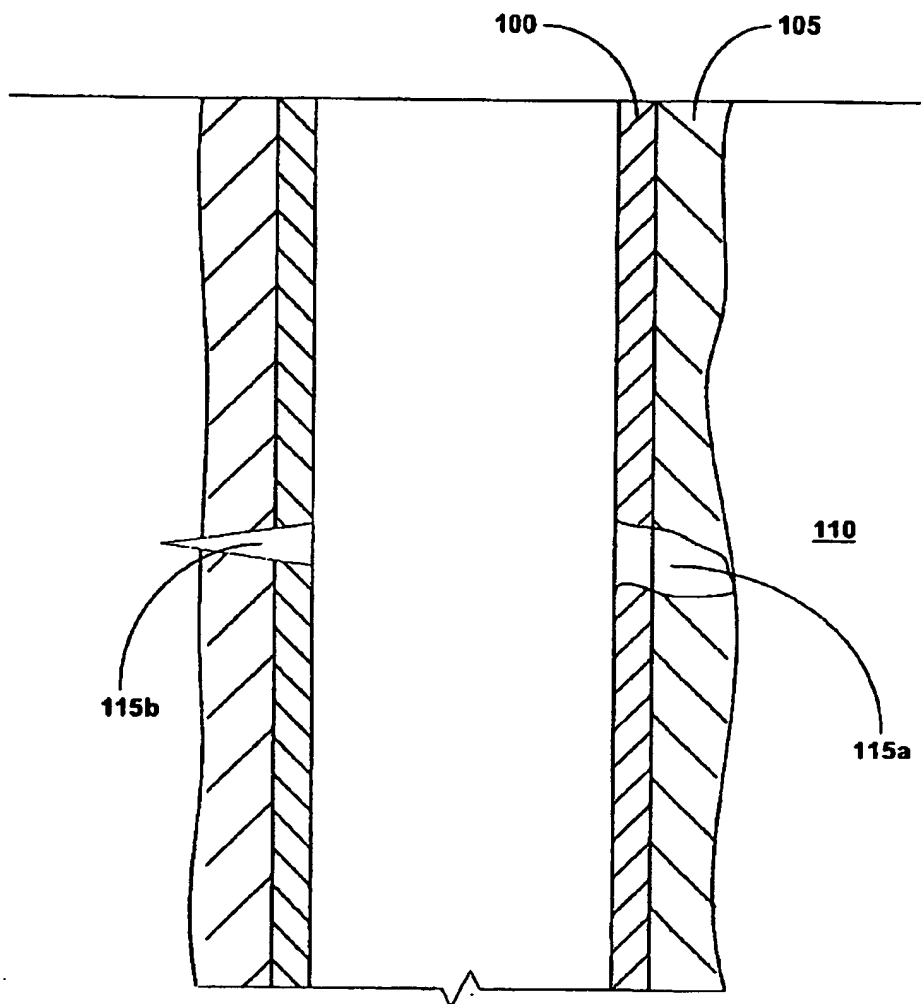


FIGURE 1

2/30

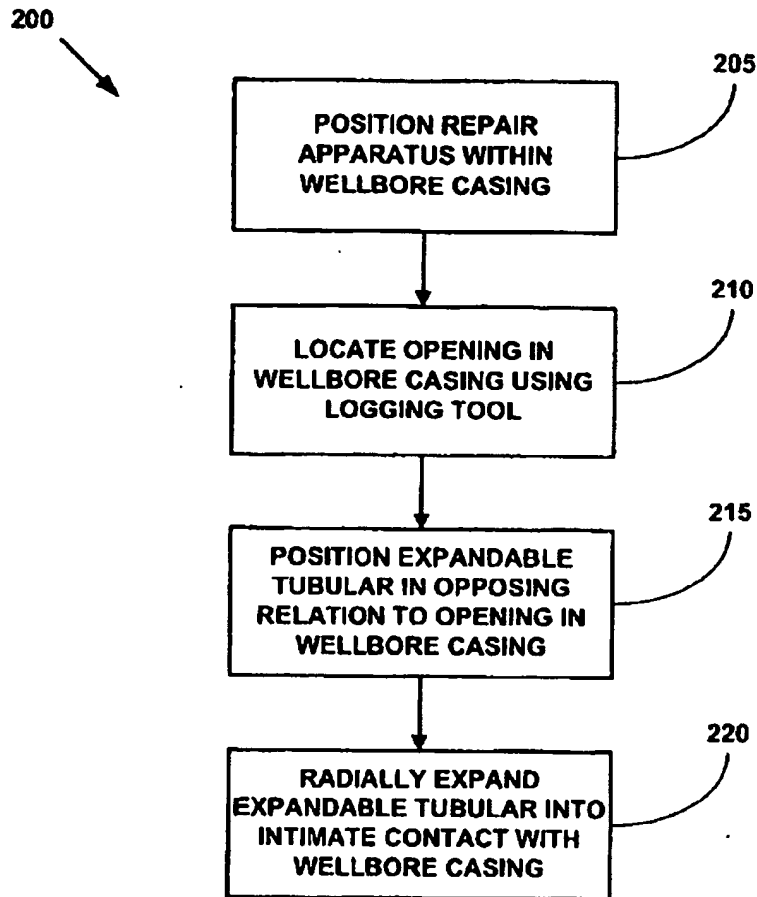
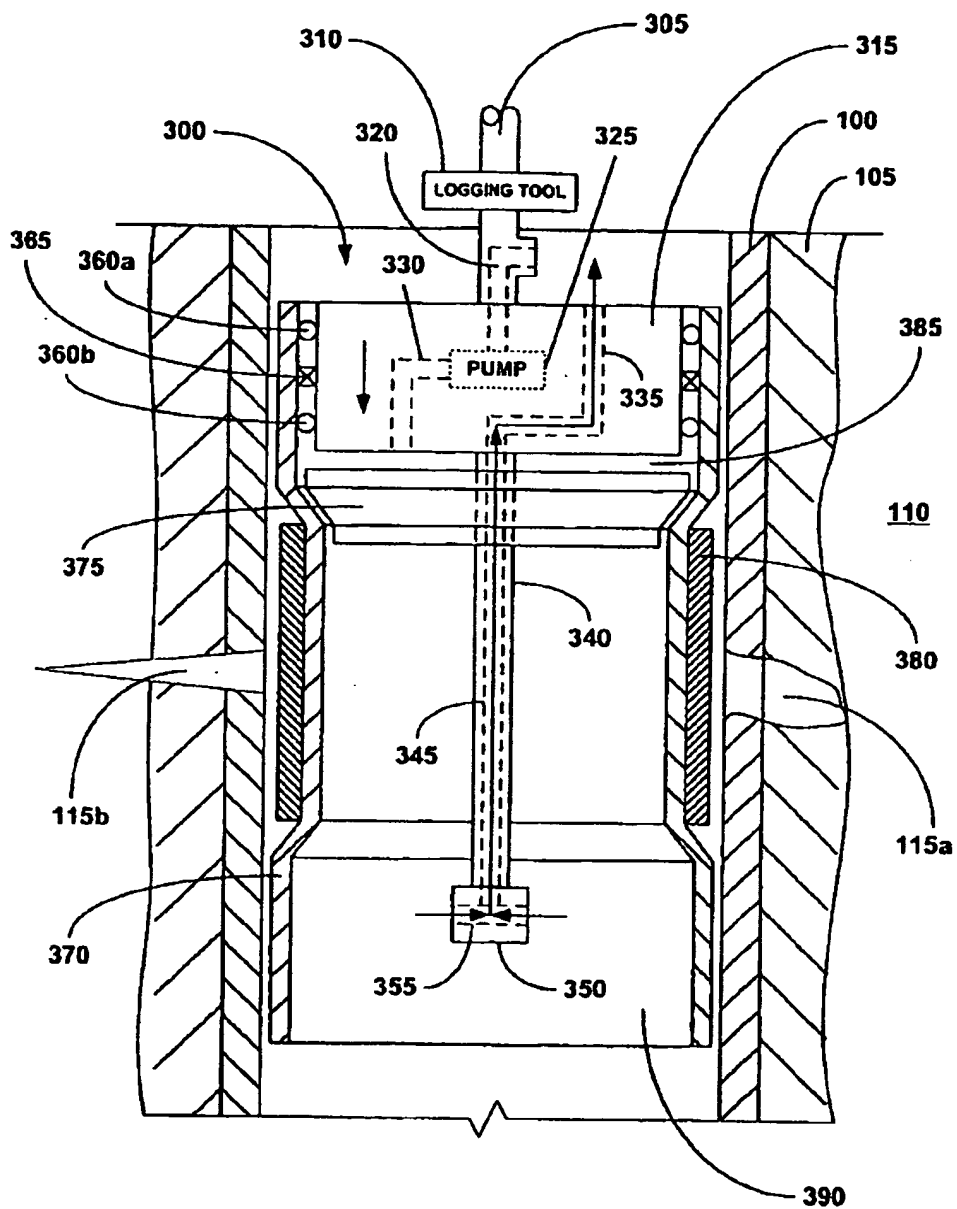


FIGURE 2

**3/30**



4/30

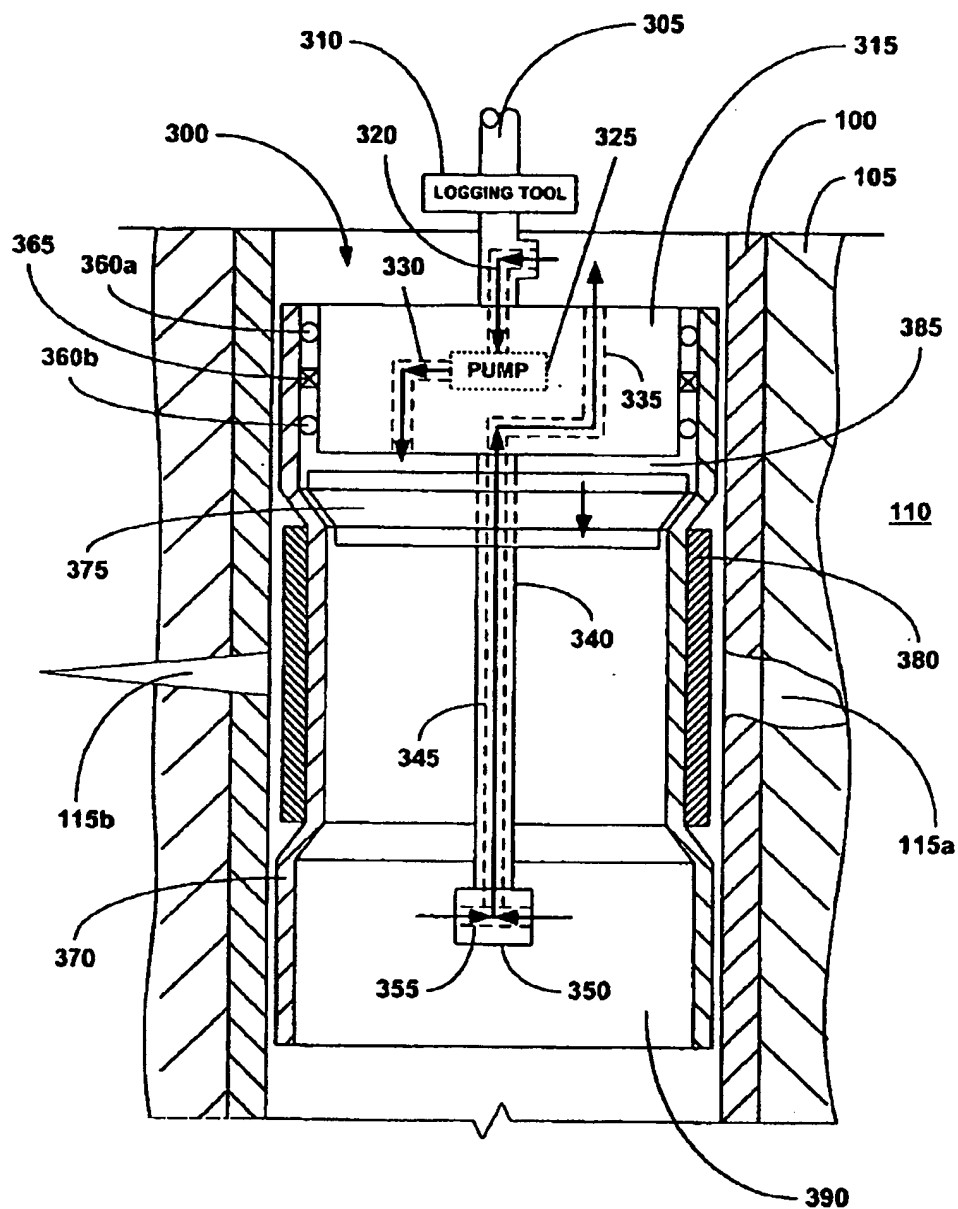
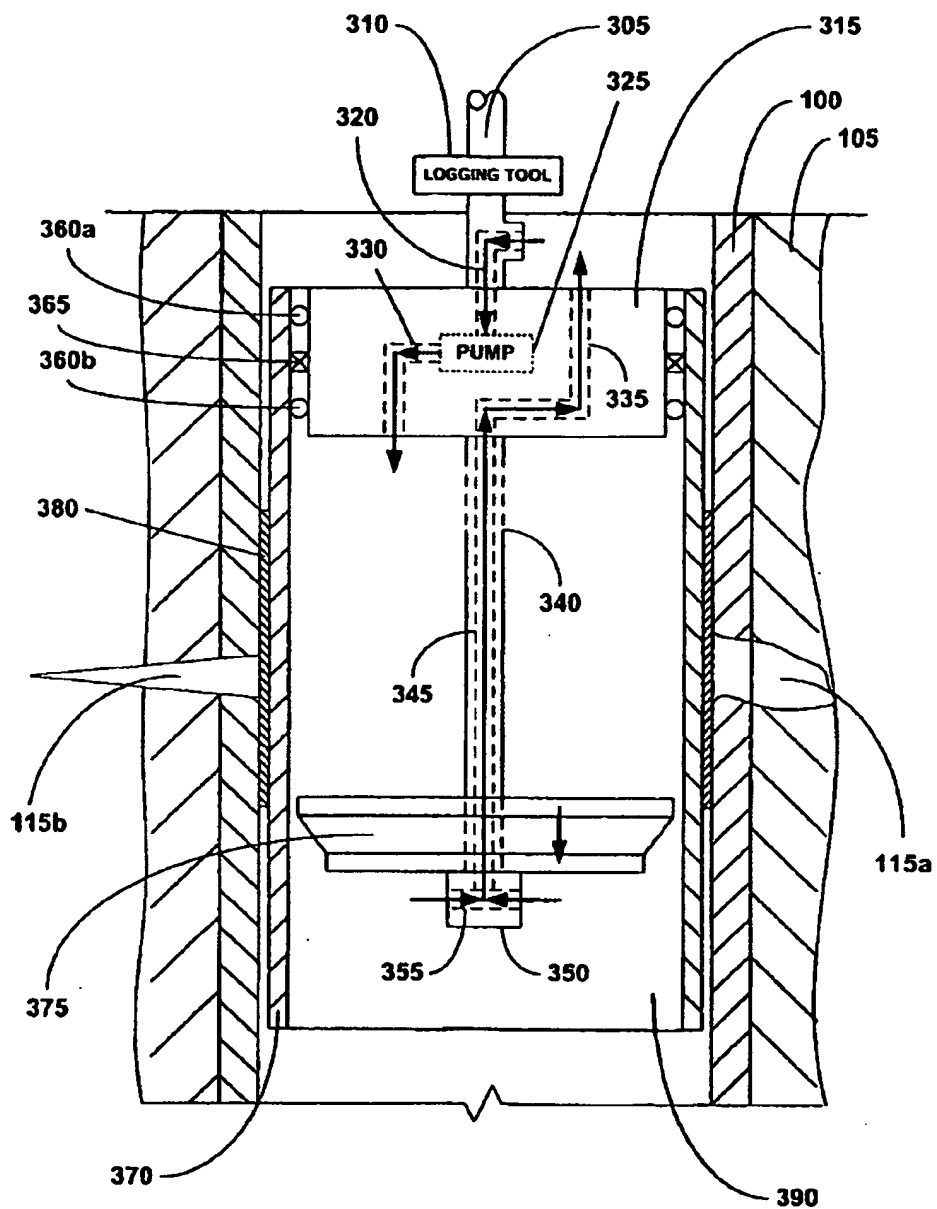


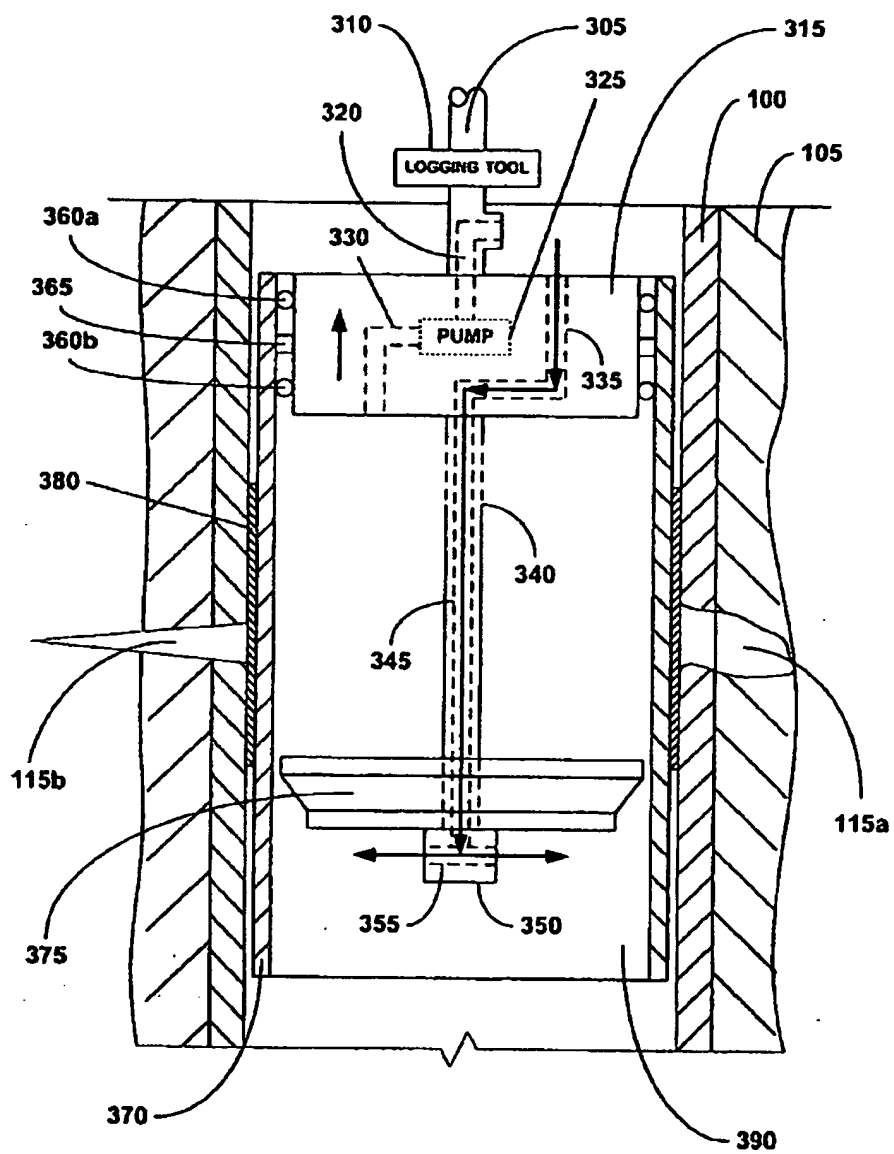
FIGURE 3b





**FIGURE 3c**

**6/30**



7/30

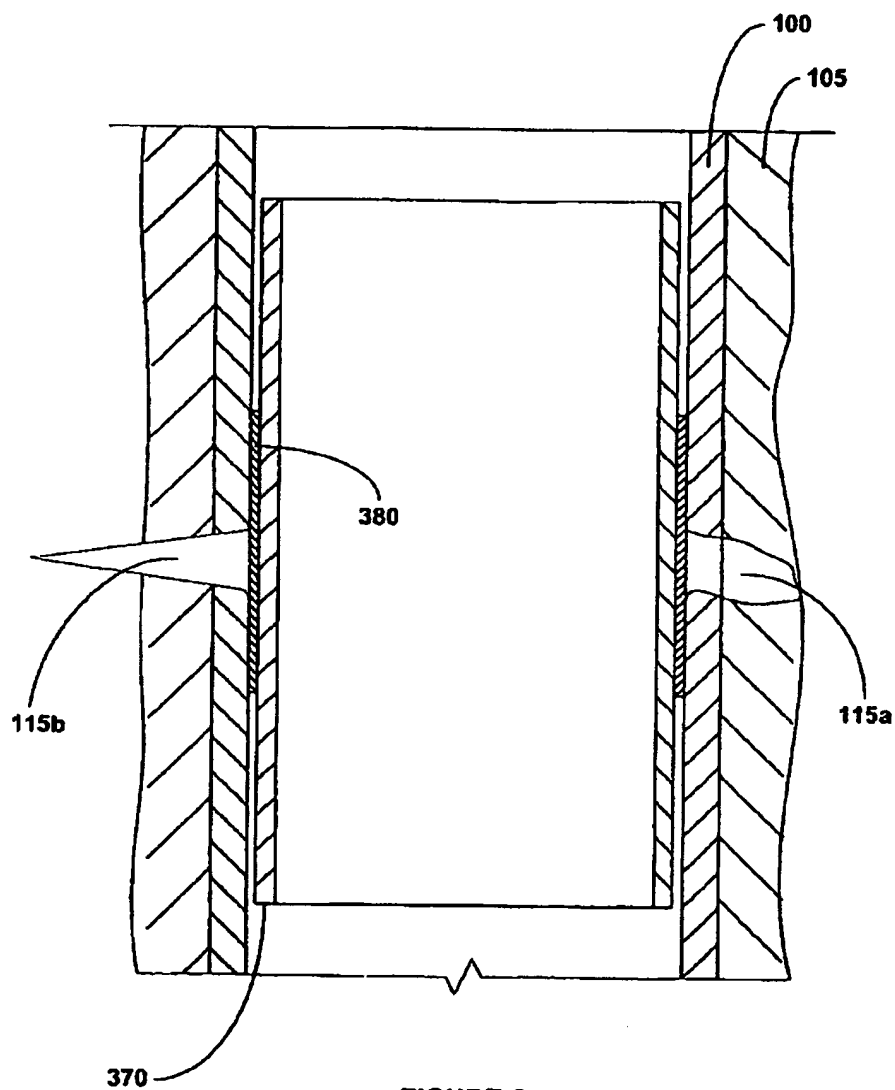
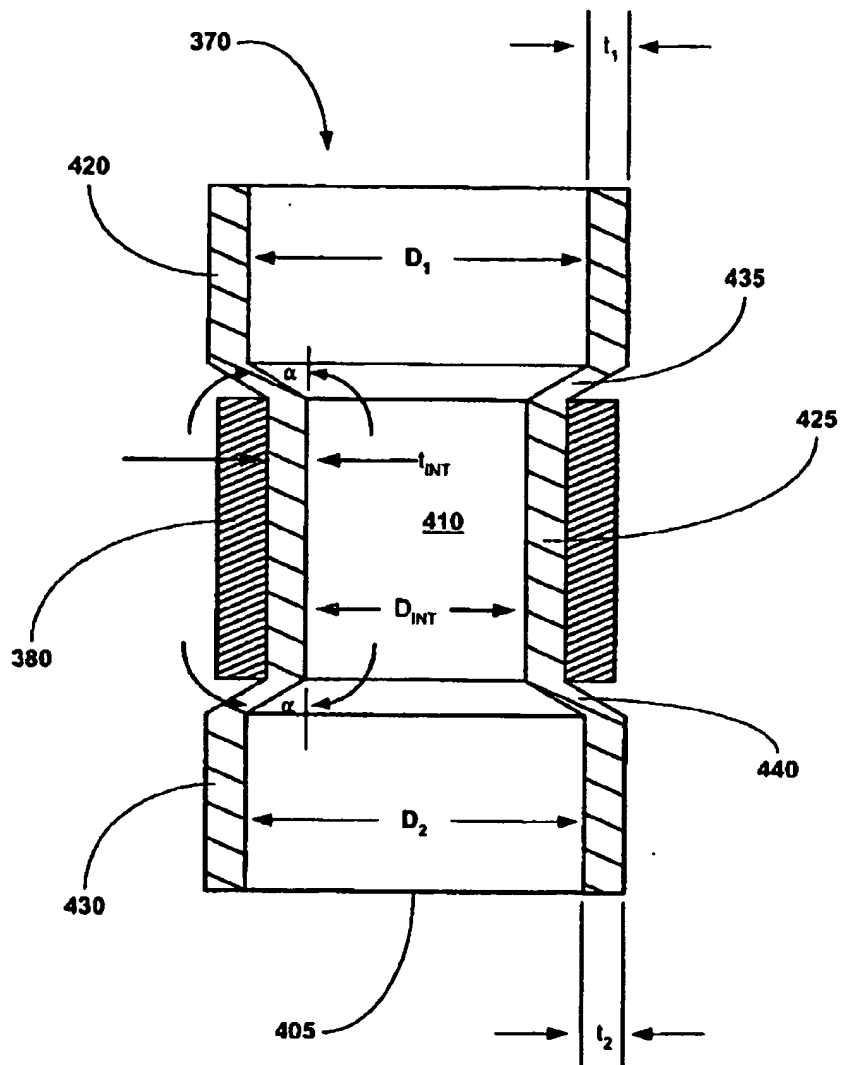


FIGURE 3e

**8/30**

**FIGURE 4**

9/30

500

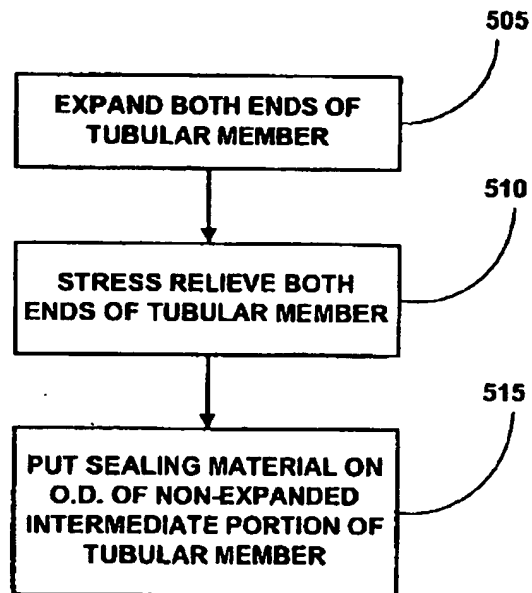


FIGURE 5



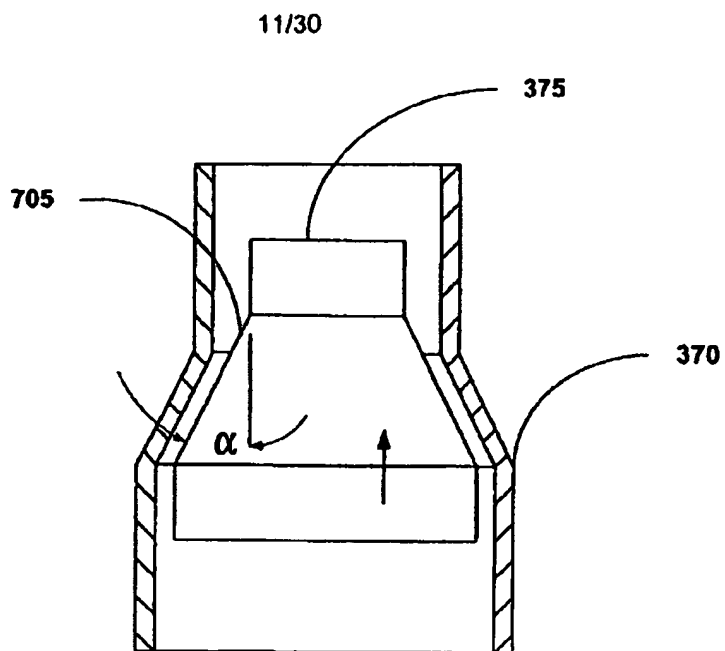


FIGURE 7

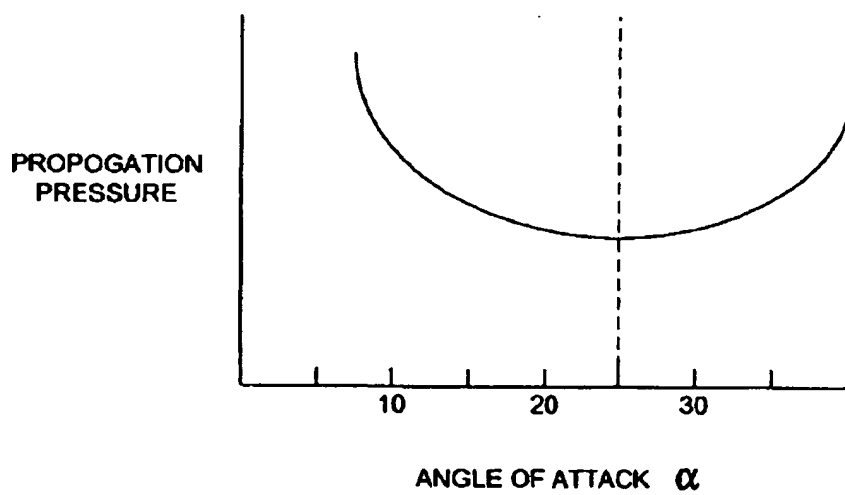


FIGURE 8

12/30

900

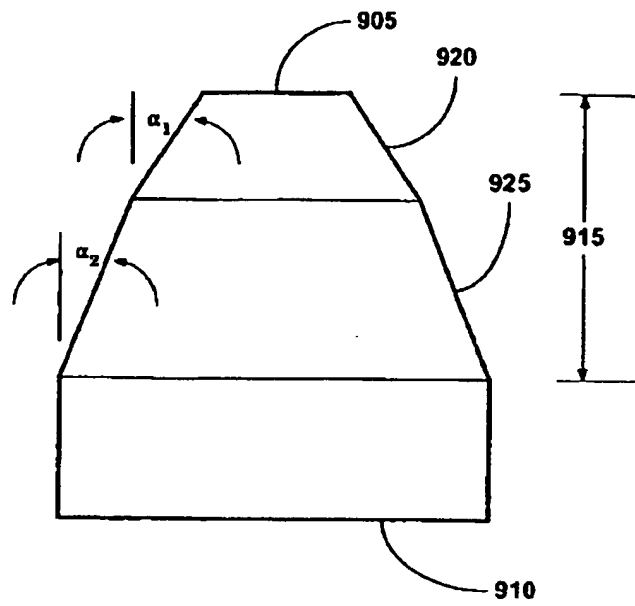


FIGURE 9

1000

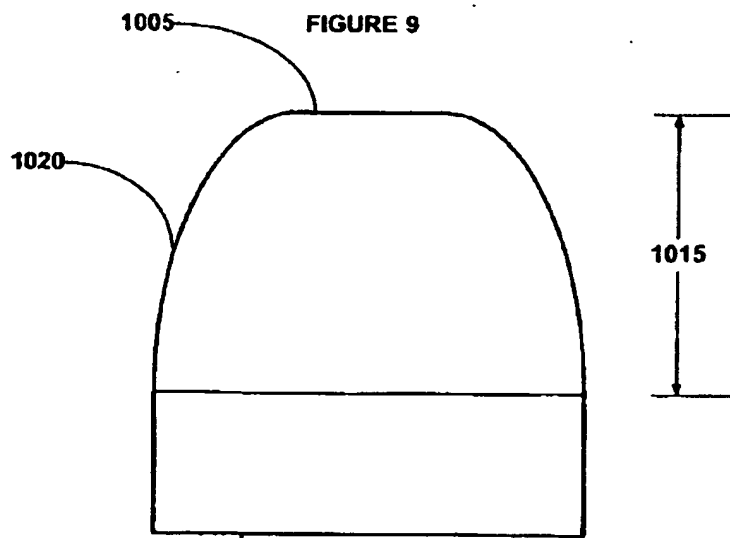


FIGURE 10

1010



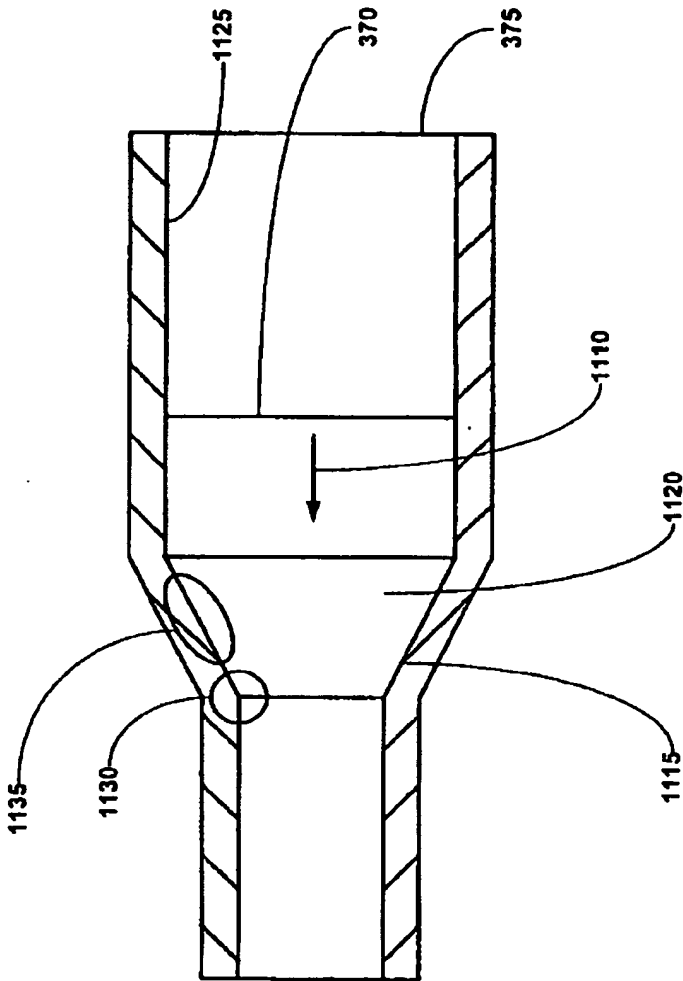
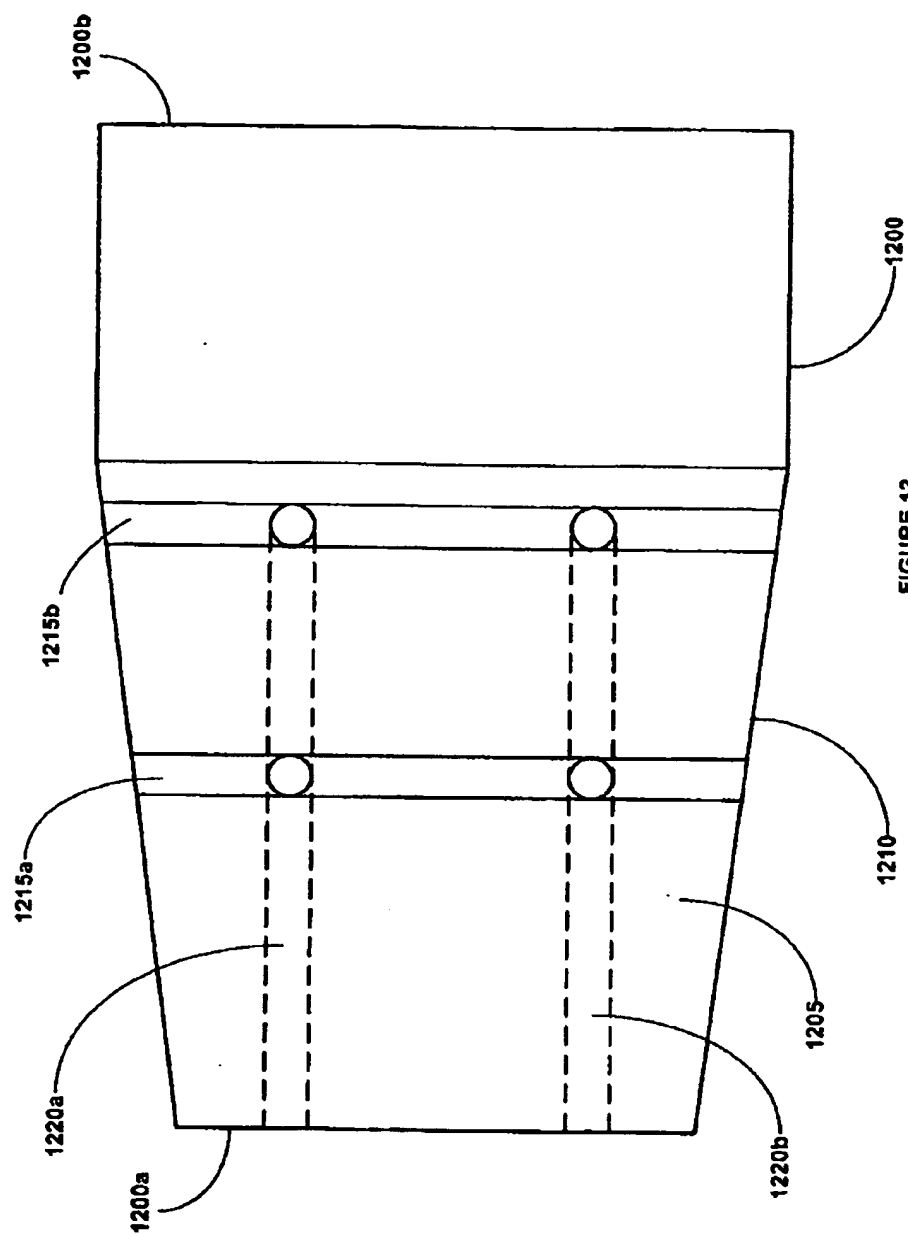


FIGURE 11



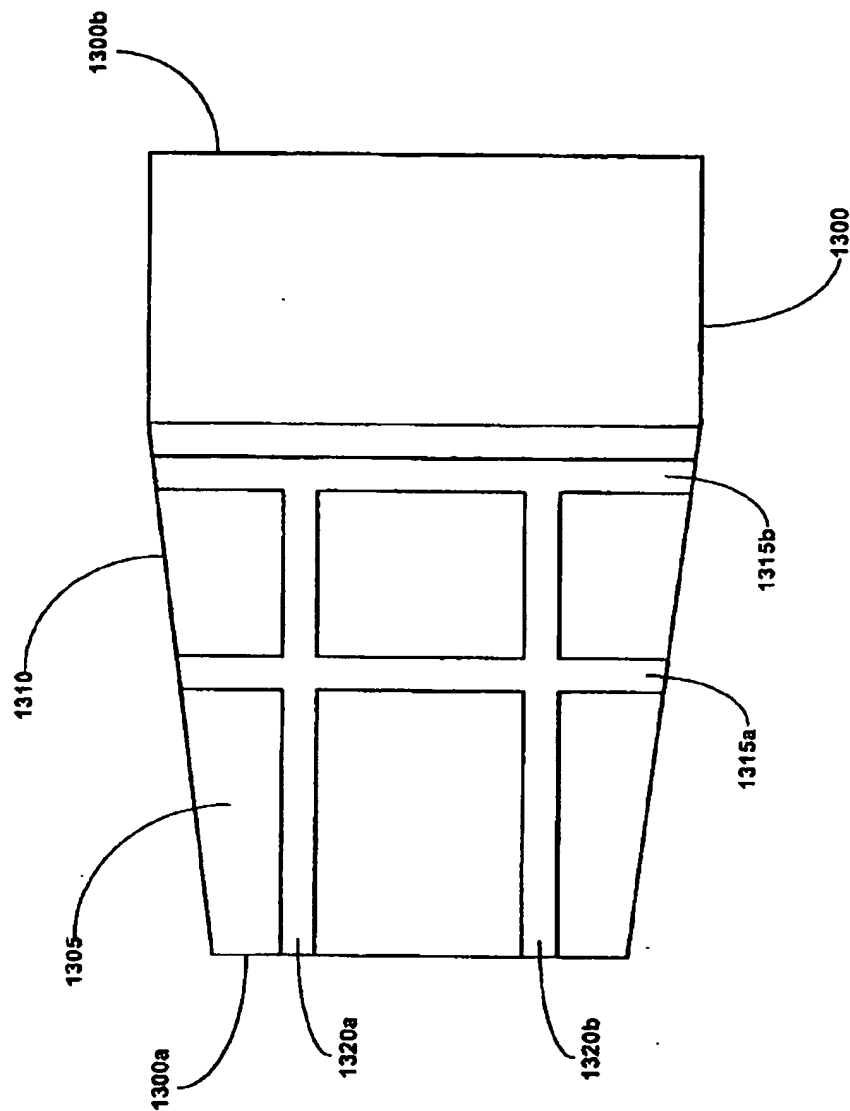
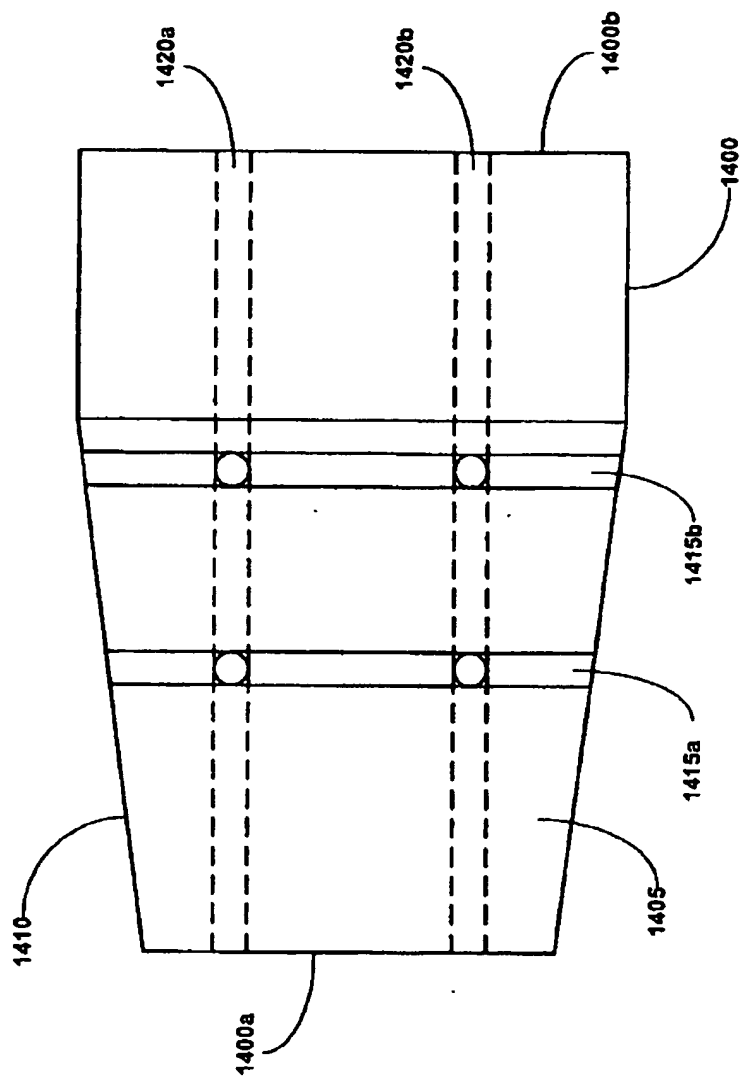


FIGURE 13



**FIGURE 14**

17/30

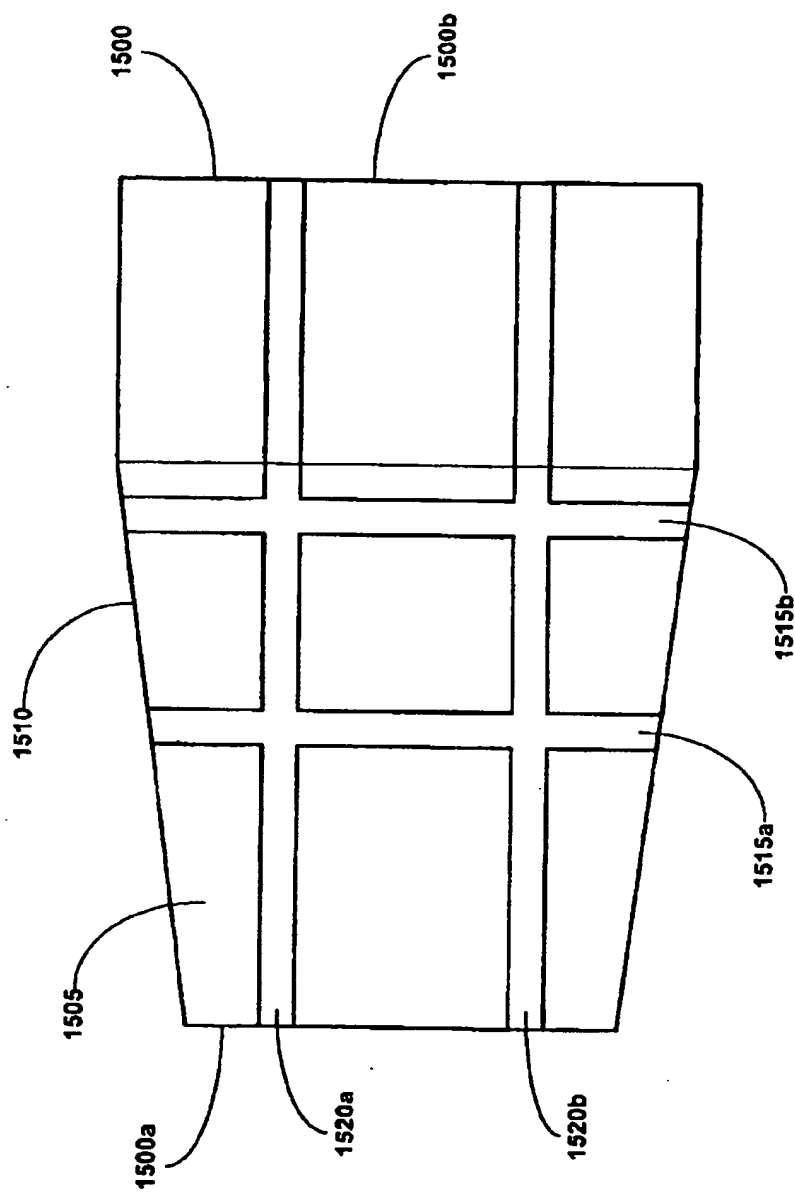


FIGURE 15

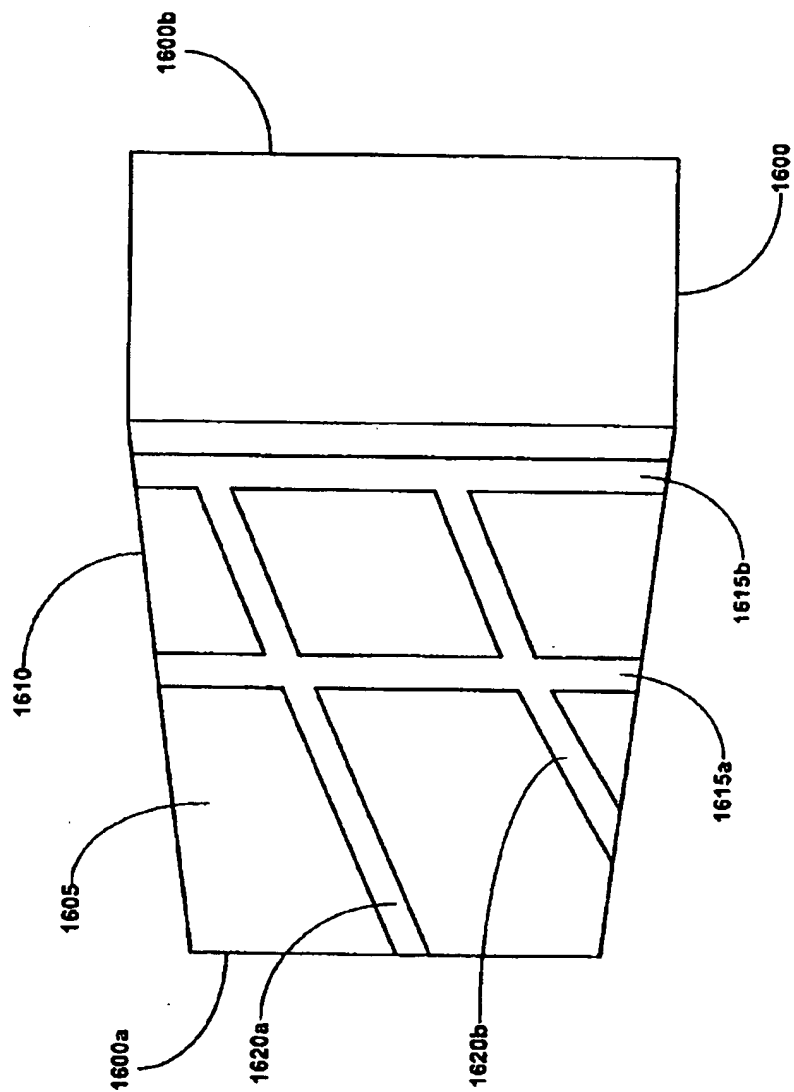


FIGURE 16

19/30

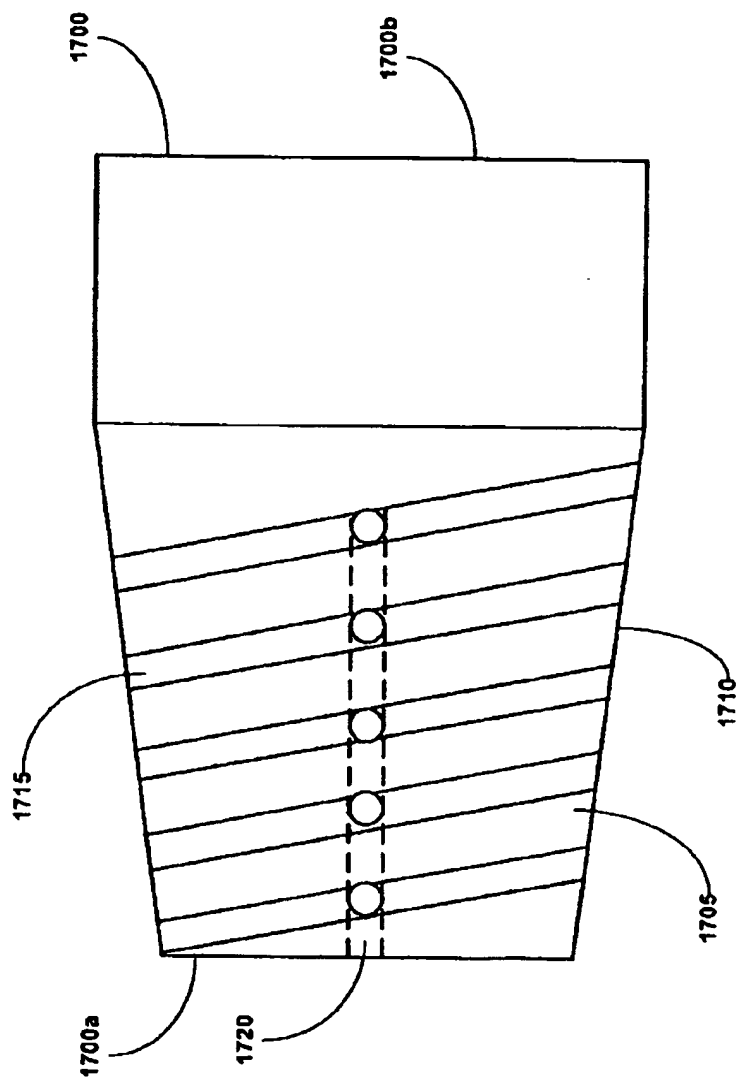


FIGURE 17

20/30

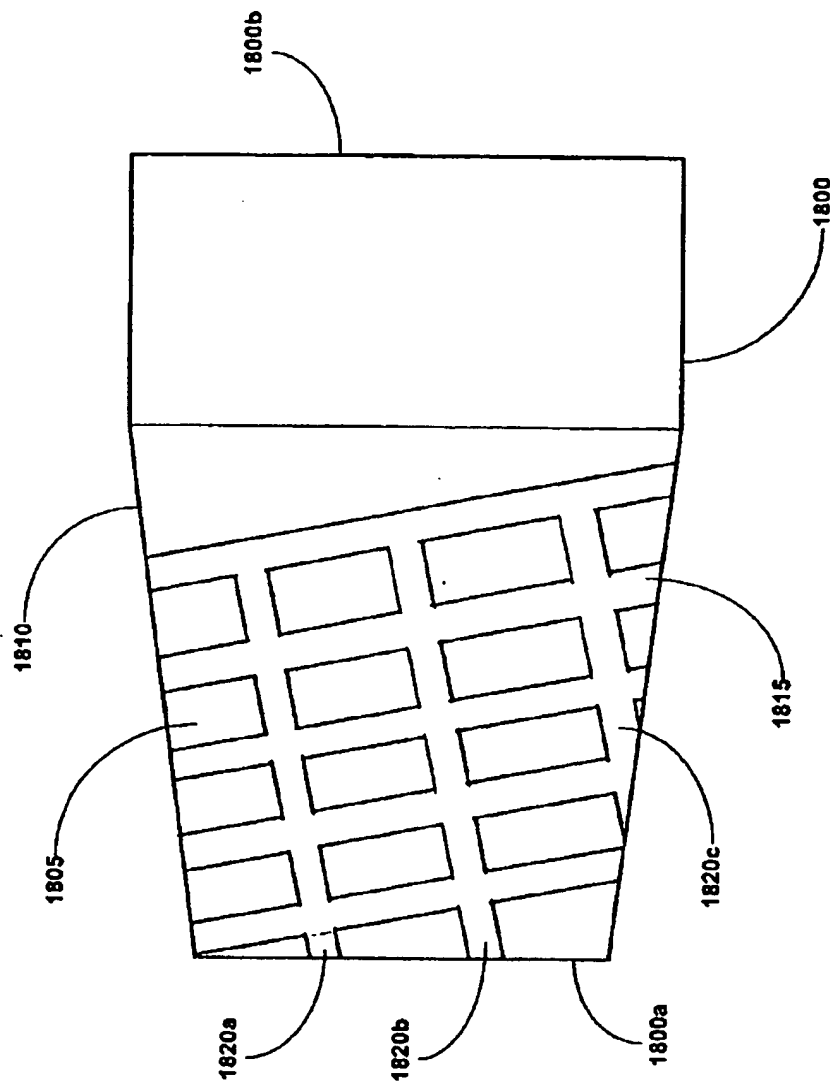
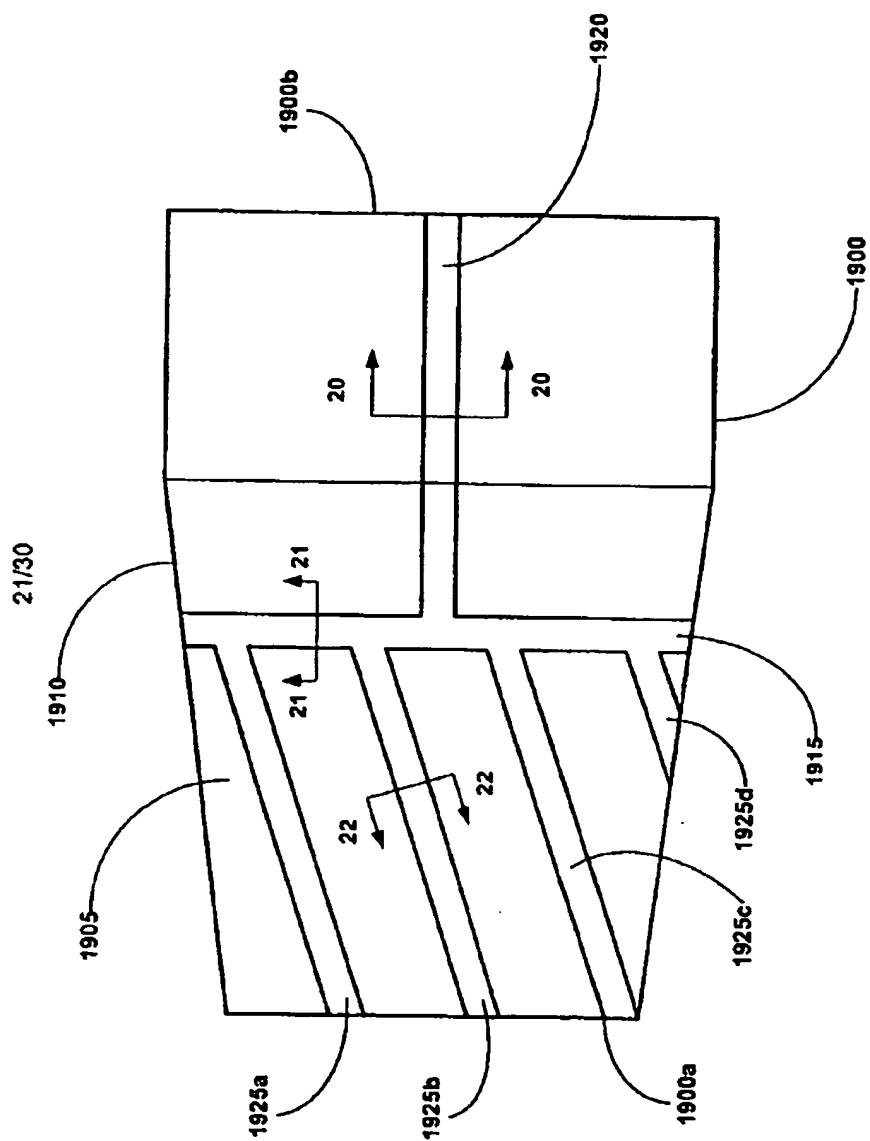


FIGURE 18





**FIGURE 19**

22/30

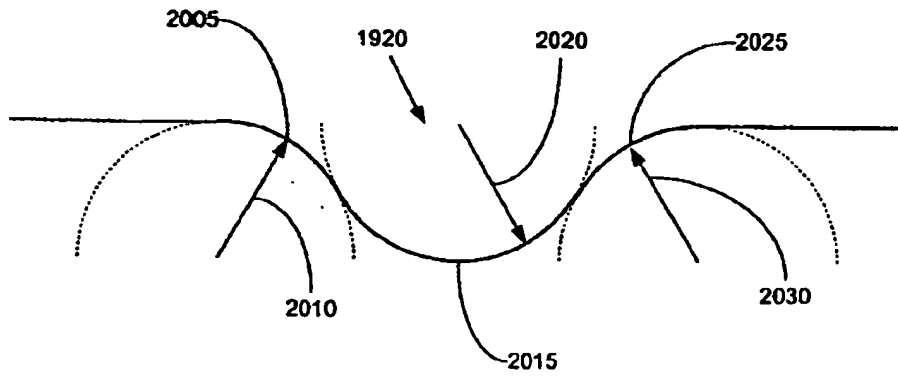


FIGURE 20

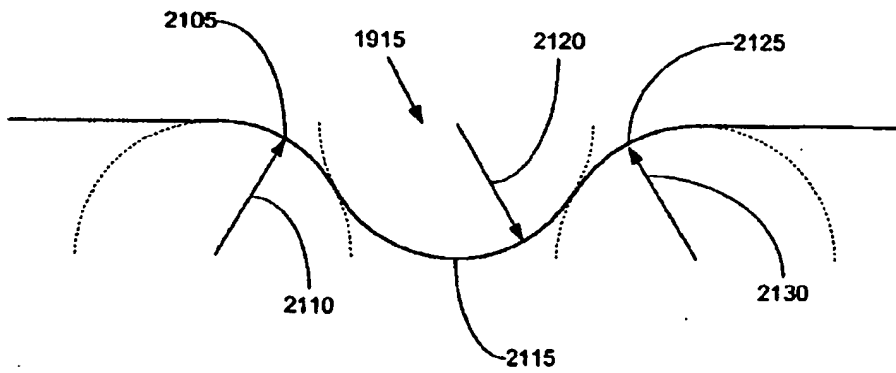


FIGURE 21

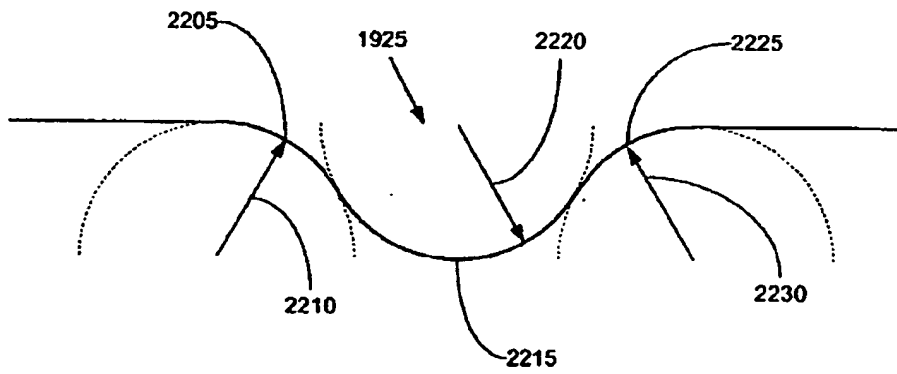


FIGURE 22

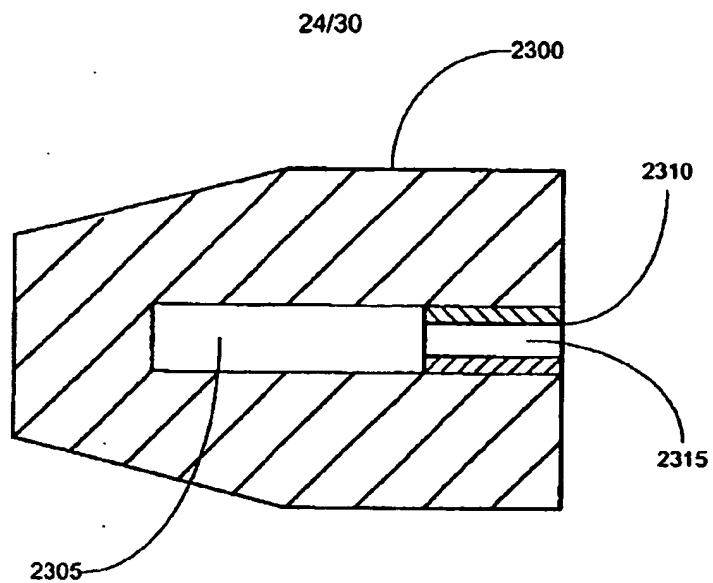


FIGURE 23

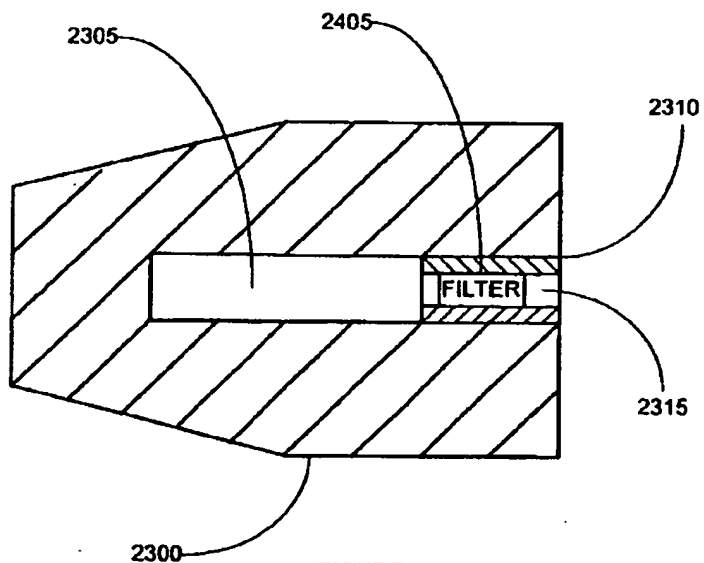


FIGURE 24

25/30

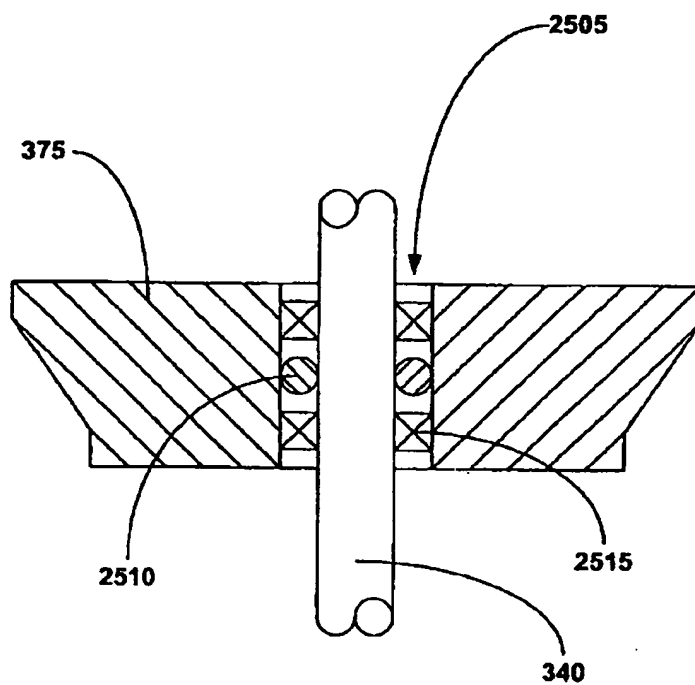


FIGURE 25



27/30

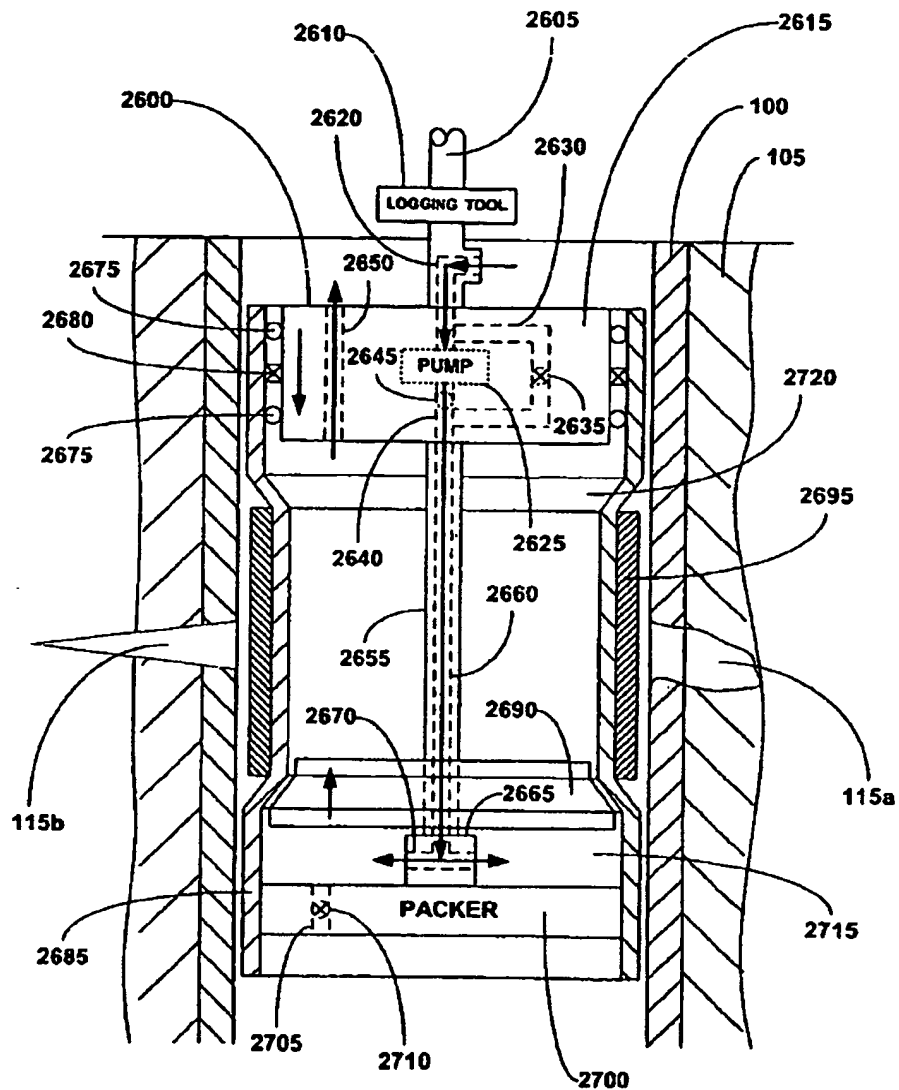


FIGURE 26b

28/30

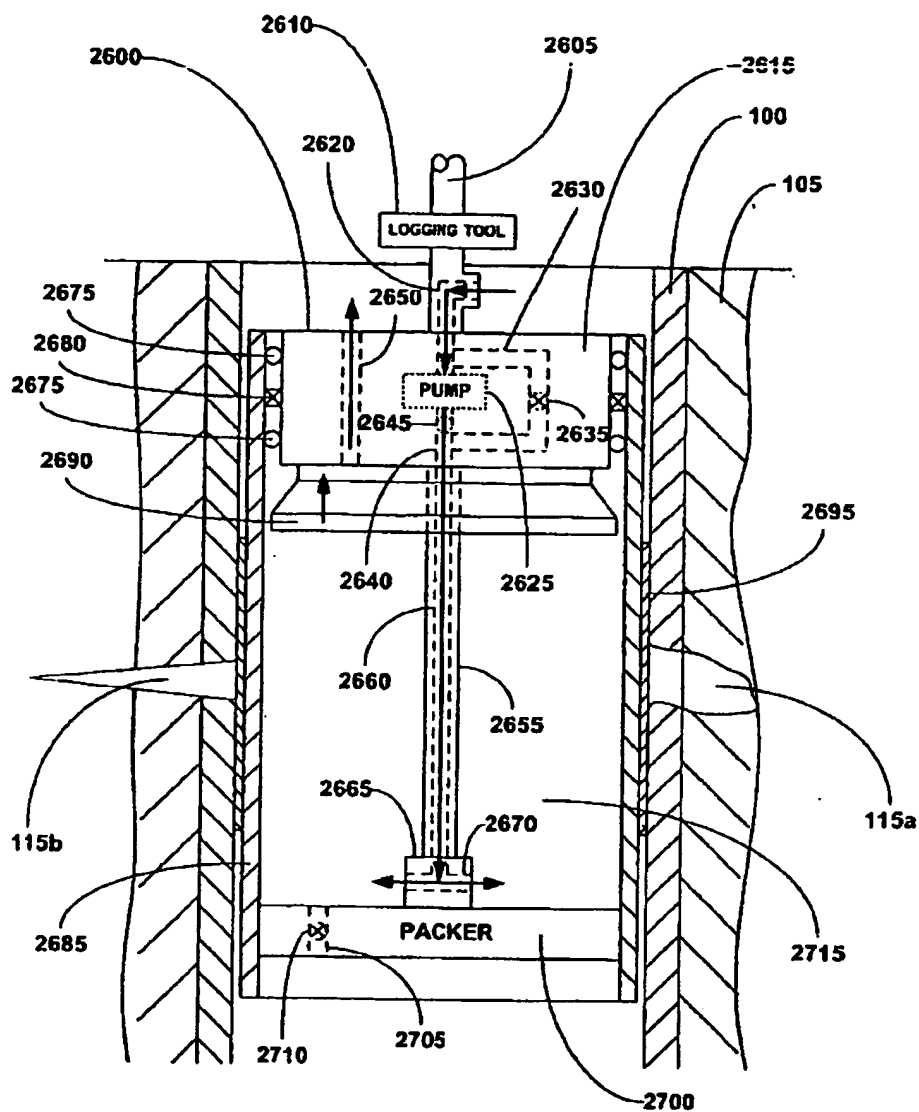
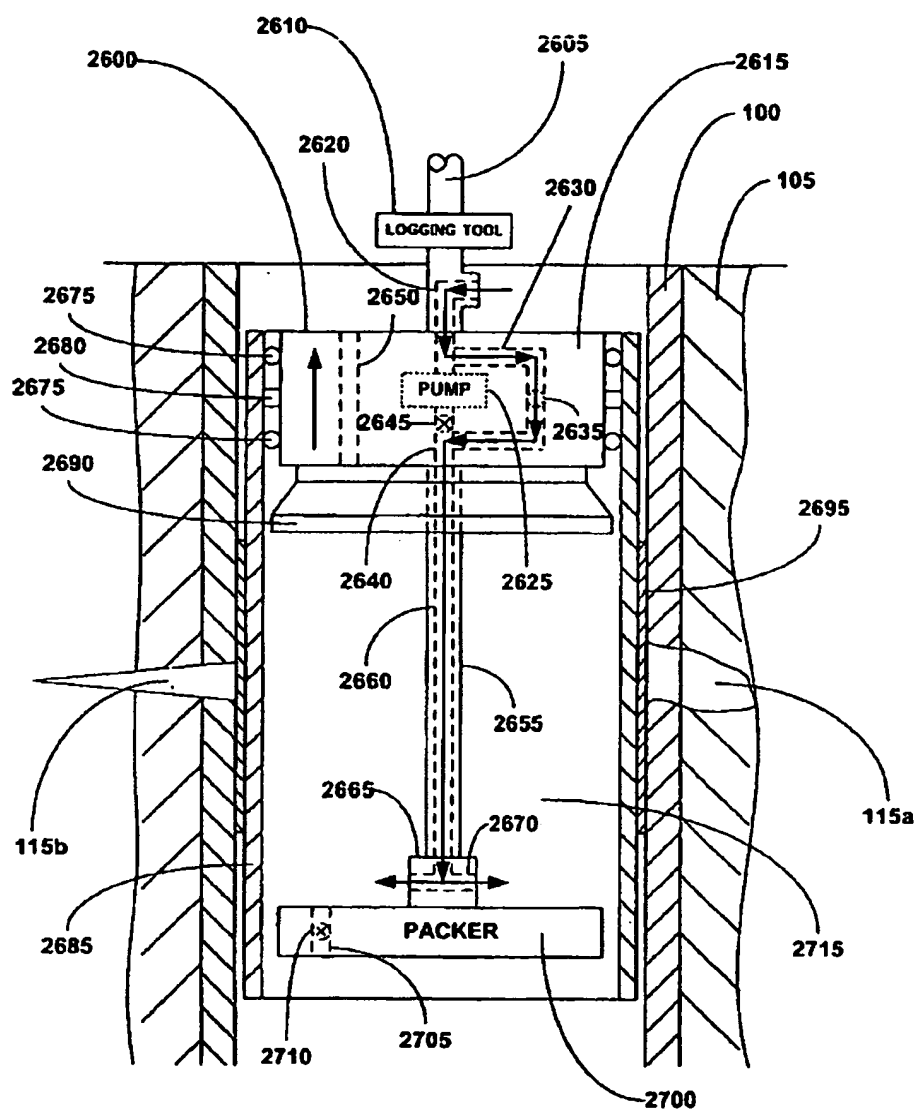


FIGURE 26c



29/30



**FIGURE 26d**

30/30

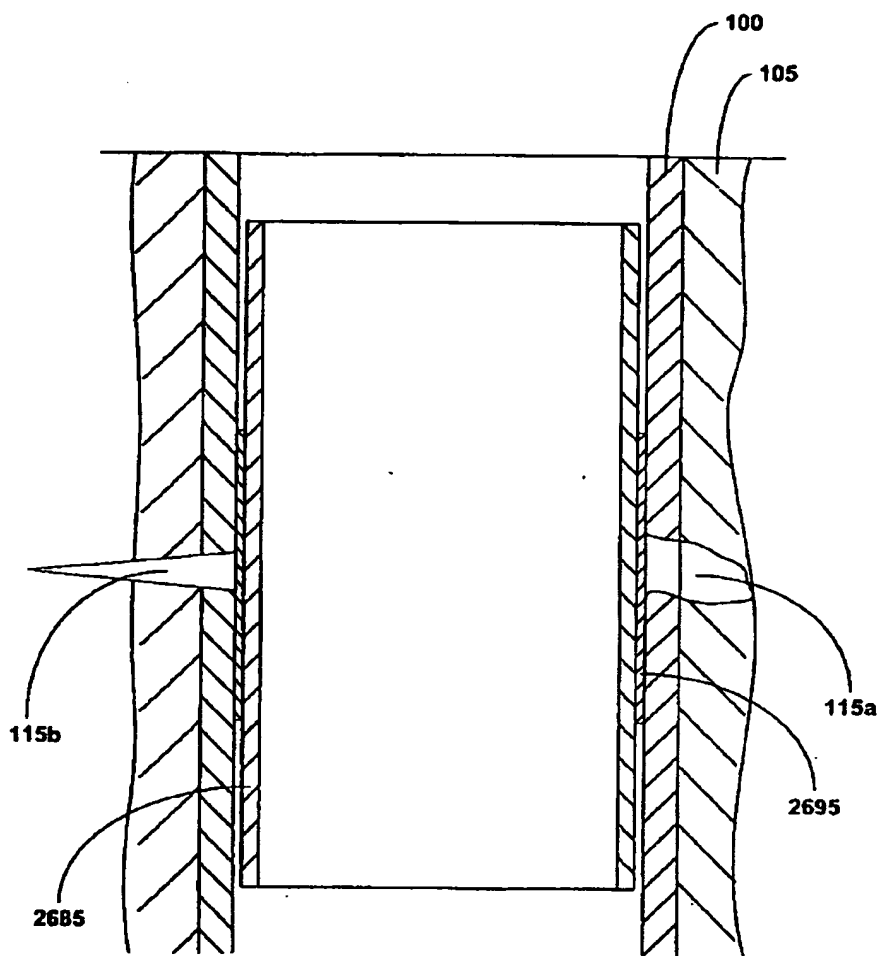


FIGURE 26e